



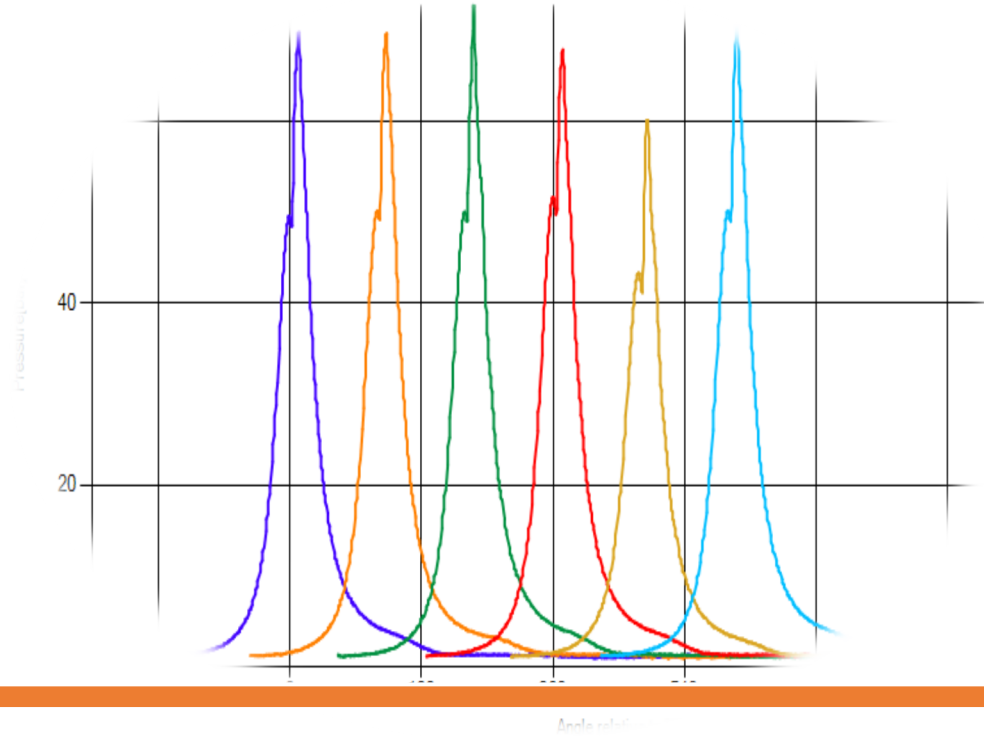
T.M.M.O.B

Gemi Makineleri İşletme Mühendisleri Odası

The Chamber of Marine Engineers

MODÜL-3

GEMİ DİZEL MOTORLARINDA PERFORMANS ANALİZİ



Prof.Dr.Adnan Parlak

MODÜL-3	GEMİ DİZEL MOTORLARINDA PERFORMANS ÖLÇÜM VE ANALİZİ – Prof. Dr. Adnan PARLAK
3.1	Fabrika Kabul Testi (FAT-Factory Acceptance Test, shop test) inceleme. Yük, Güç, Özgül Yakıt sarfiyatı, skavenç basıncı, türbin devri, hava kuleri çıkış sıcaklığı, hava kuleri fark basınç, yüke bağlı kompresyon ve yanma basınçlarının incelenmesi, yüke bağlı egzoz sıcaklık ilişkilerinin analizi.
3.2	Silindir içi performans ölçümü ve ölçüm sonuçlarının analiz edilmesi. Bu kapsamda yanma basınçlarının krank açısına göre değişimlerinin analizi, silindirler arası kompresyon basınç değişimlerinin analizi, Pmax-Pcomp fark analizi, silindirler arası Pmax değişim analizi, maksimum basıncın gerçekleştiği krank açısının önemi, silindirler arası güç ve yük dağılımının analiz edilmesi
3.3	Çevresel şartların yakıt sarfiyatına etkisinin ve özgül yakıt sarfiyatının ISO 3046 şartlarına indirgenmesi
3.4	Ölçülmüş performans grafikleri üzerinden hata analizi ve makinelerin fabrika ayarlarına geri getirmek için çözüm önerileri
3.5	Atölye çalışması (Kursiyerler 4 ya da 5 li gruba ayrılarak kendilerine verilen farklı ölçüm değerleri için çözüm önerisi içerir hazırlayacak ve sunacaklar)

Yakıt Tüketimi (FC) : Belirli bir yük ve devirde içten yanmalı bir motorun saatte tükettiği yakıt miktarıdır. Birim kg/h

Özgül Yakıt Tüketimi (SFC) : Bir içten yanmalı motorun kWh başına tükettiği yakıt miktarını ifade eder. Birimi g/kWh

Fren veya Efektif Güç (kW) : İçten yanmalı motorun fren dinamometresi ile ölçülen güç değeridir. Fren gücü, silindirde üretilen belirtilen güçten sürtünme gücünün çıkarılmasıyla bulunur. Birimi kW

Alt Isıl Değeri (LCV): Bir yakıtın ısı değeri, yanması sırasında salınan ısı miktarıdır. Enerji veya kalori değeri olarak da adlandırılan ısı değeri, bir yakıtın enerji yoğunluğunun bir ölçüsüdür ve belirli bir miktar (ör. kilogram) başına enerji (joule) olarak ifade edilir. $kJ/kg, MJ/kg$

Ortalama İndike Basınç /Mean Indicated Pressure (MIP):Silindire içerisinde bir çevrim boyunca oluşan basınçların ortalaması olarak ifade edilir.

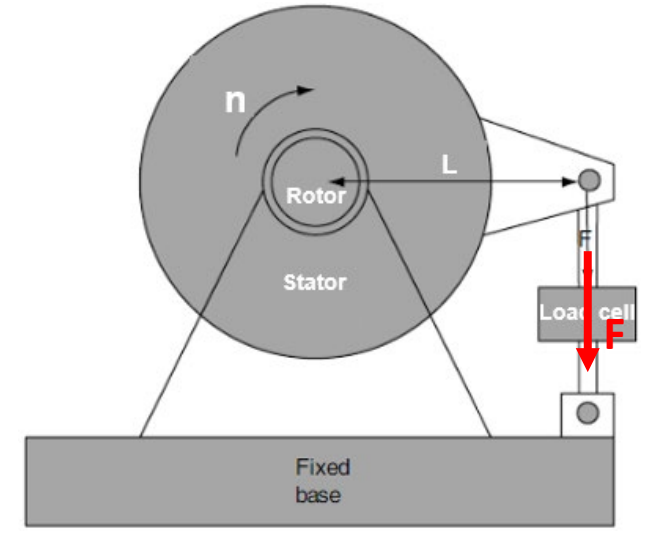
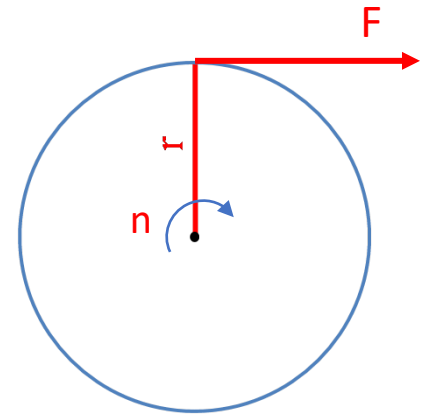
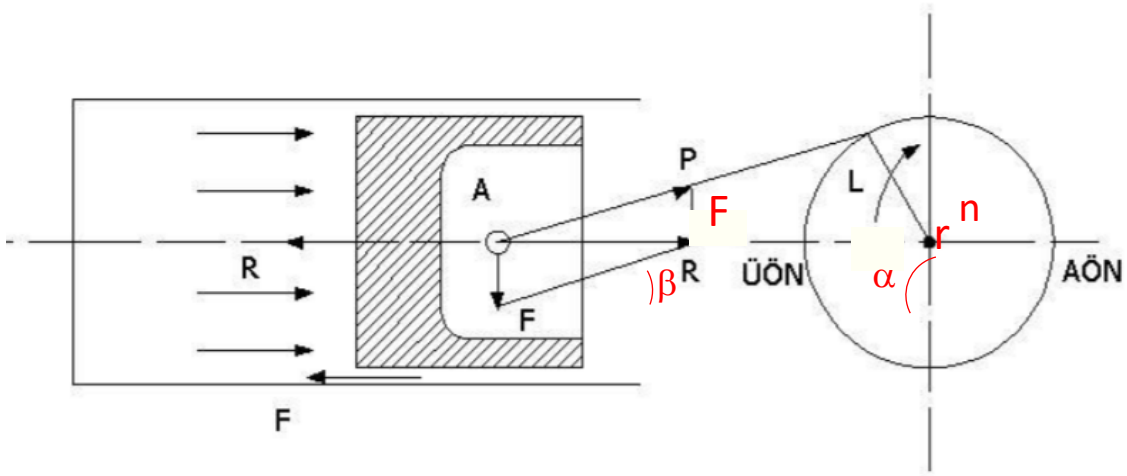
Moment (Tork): Bir cismi bir eksen etrafında döndürme etkisi, Nm

Güç/Power: Birim zamanda gerçekleşen iş sayısı, kW



Fren yada Efektif Güç

Brake Power Formula



Engine Torque = Brake Torque

$$M_d = F \cdot r$$

$$M_{brake} = F \cdot L$$

Engine Work

Brake Work L=0,955 m
F → N

$$W_e = 2\pi Fr = F \cdot (2\pi r)$$

$$W_b = 2\pi FL$$

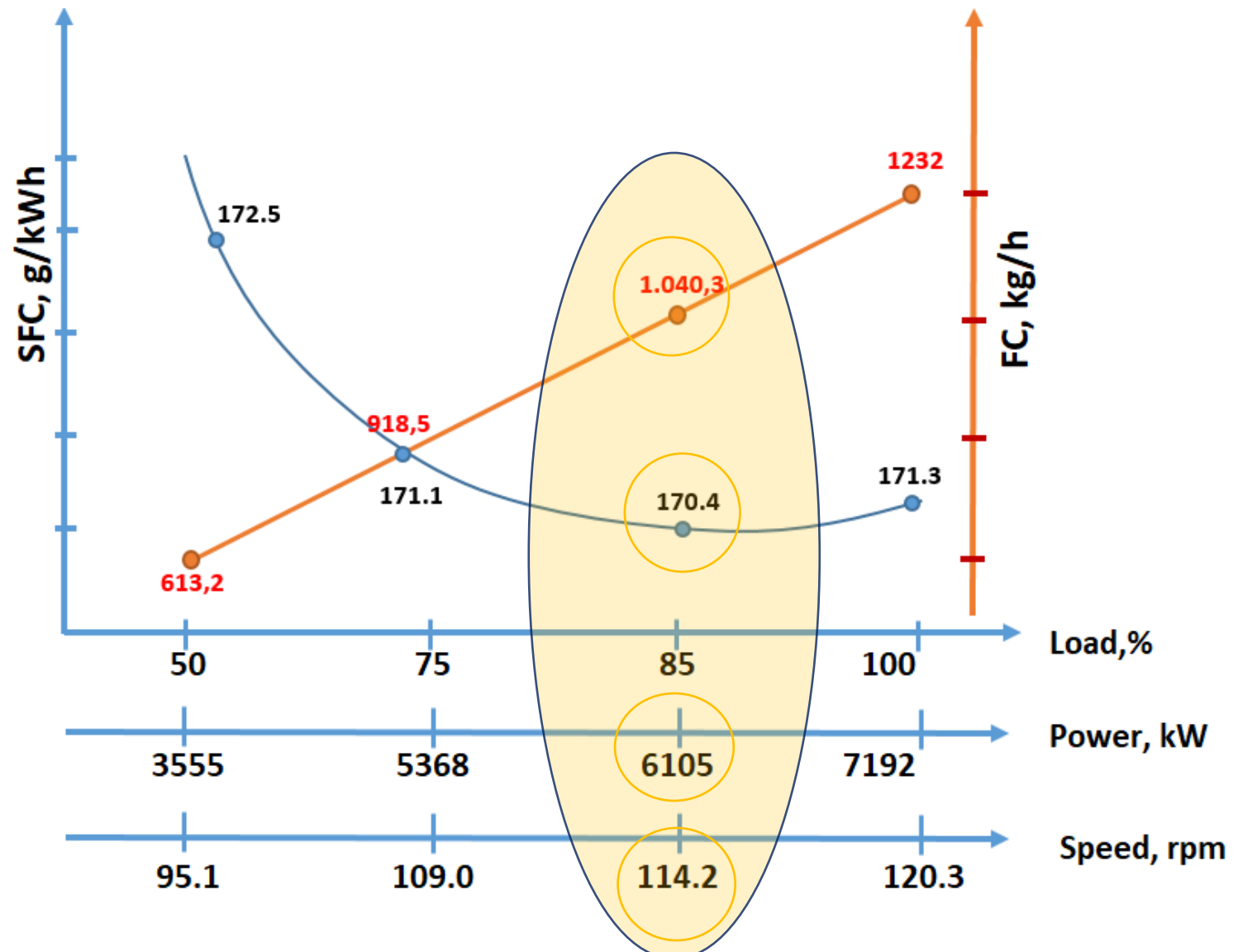
Engine Power

Brake Power

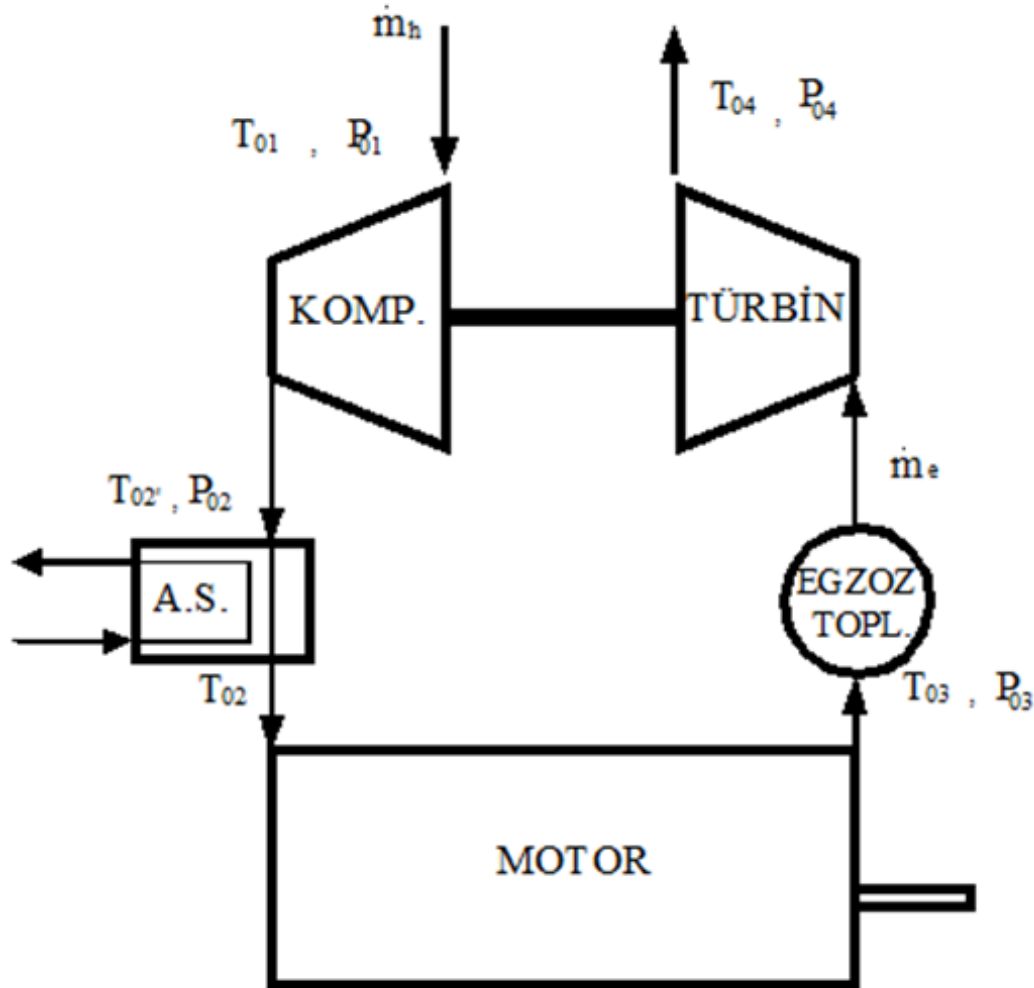
$$\dot{W}_e = 2\pi n Fr = \omega \cdot M_d$$

$$W_b = 2\pi FLn = \frac{F \cdot n}{10^4} \quad [\text{kW}]$$

Engine Performance Curve (Shop Test)



Türbin Giriş-Çıkış Sıcaklığı ve Skavenç Basınç ilişkisi



Türbin ve Kompresör gücü arasındaki ilişki:

$$P_K = \eta_m \cdot P_T$$

Kompresör için gerekli olan güç

$$P_K = \frac{m_h \cdot c_p \cdot (T_{02} - T_{01})}{\eta_K} = \frac{m_h \cdot c_p \cdot T_{01}}{\eta_K} \left[\left(\frac{P_{02}}{P_{01}} \right)^{\frac{k-1}{k}} - 1 \right]$$

Türbin gücü ise

$$P_T = m_e \cdot c_{p,e} \cdot \eta_T \cdot (T_{03} - T_{04})$$

Türbin ve kompresör arasındaki ilişki

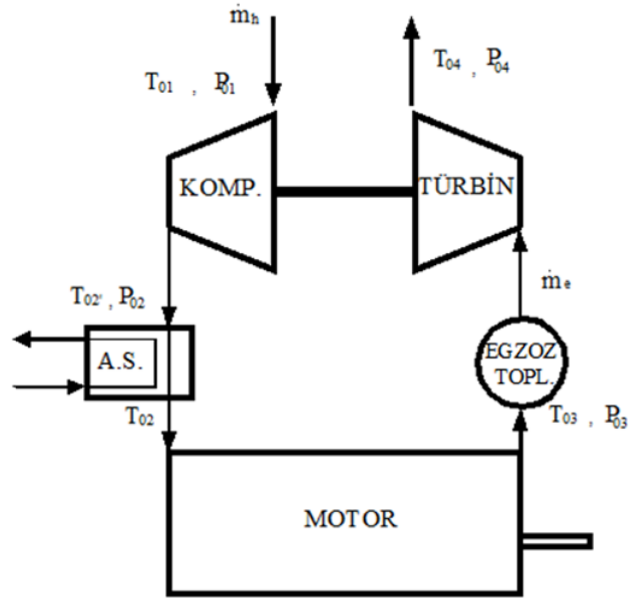
$$\dot{m}_e \cdot c_{p,e} \cdot (T_5 - T_6) \eta_T \cdot \eta_K \cdot \eta_m = \dot{m}_h \cdot c_{p,h} \cdot (T_2 - T_1)$$

Kompresör çıkış sıcaklığı:

$$T_2 = T_1 + \frac{\dot{m}_e \cdot c_{p,e} \cdot (T_5 - T_6) \eta_T \cdot \eta_K \cdot \eta_m}{\dot{m}_h \cdot c_{p,h}}$$

Kompresör çıkış basıncı:

$$P_{02} = P_{01} \left(\frac{T_{02}}{T_{01}} \right)^{\frac{k}{k-1}}$$



k	1,4
T ₀₁ , C	25
p ₀₁ kPa	100
T ₀₁ , C	25
T ₀₃ , C	494
T ₀₄ , C	386
η _T	0,9
η _K	0,9
η _m	0,9
C _{p,a} kJ/kgK	1,004
C _{p,e} kJ/kgK	1,044
m _h , kg/h	974
m _v , kg/h	16,6
m _e , kg/h	990,6

Kompresör çıkış Sıcaklığı, K	381,3	108,3
Skavenç Basıncı, kPa	236,88	kPa
Gauge Pressure, kPa	136,88	kPa

k	1,4
T ₀₁ , C	25
p ₀₁ kPa	100
T ₀₁ , C	25
T ₀₃ , C	494
T ₀₄ , C	400
η _T	0,9
η _K	0,9
η _m	0,9
C _{p,a} kJ/kgK	1,004
C _{p,e} kJ/kgK	1,044
m _h , kg/h	974
m _v , kg/h	16,6
m _e , kg/h	990,6

Kompresör çıkış Sıcaklığı, K	370,5	97,5
Skavenç Basıncı, kPa	214,23	kPa
Gauge Pressure, kPa	114,23	kPa

k	1,4
T ₀₁ , C	25
p ₀₁ kPa	100
T ₀₁ , C	25
T ₀₃ , C	494
T ₀₄ , C	425
η _T	0,9
η _K	0,9
η _m	0,9
C _{p,a} kJ/kgK	1,004
C _{p,e} kJ/kgK	1,044
m _h , kg/h	974
m _v , kg/h	16,6
m _e , kg/h	990,6

Kompresör çıkış Sıcaklığı, K	351,2	78,2
Skavenç Basıncı, kPa	177,69	kPa
Gauge Pressure, kPa	77,69	kPa

Efektif Güç

Moment

Ortalama Basınç

$$P_e = \frac{\eta_e \cdot \eta_v \cdot V_h \cdot L \cdot H_u \cdot \rho_{h,g}}{2} \cdot (Y/H)$$

$$T = \frac{\eta_e \cdot \eta_v \cdot V_h \cdot H_u \cdot \rho_{h,g}}{4\pi} \cdot (Y/H)$$

$$P_{me} = \eta_e \cdot \eta_v \cdot H_u \cdot \rho_{h,g} \cdot (Y/H)$$

Ortalama Basınca göre güç hesabı

$$P_e = \frac{p_{me} \cdot V_H \cdot n \cdot i}{60.000}$$



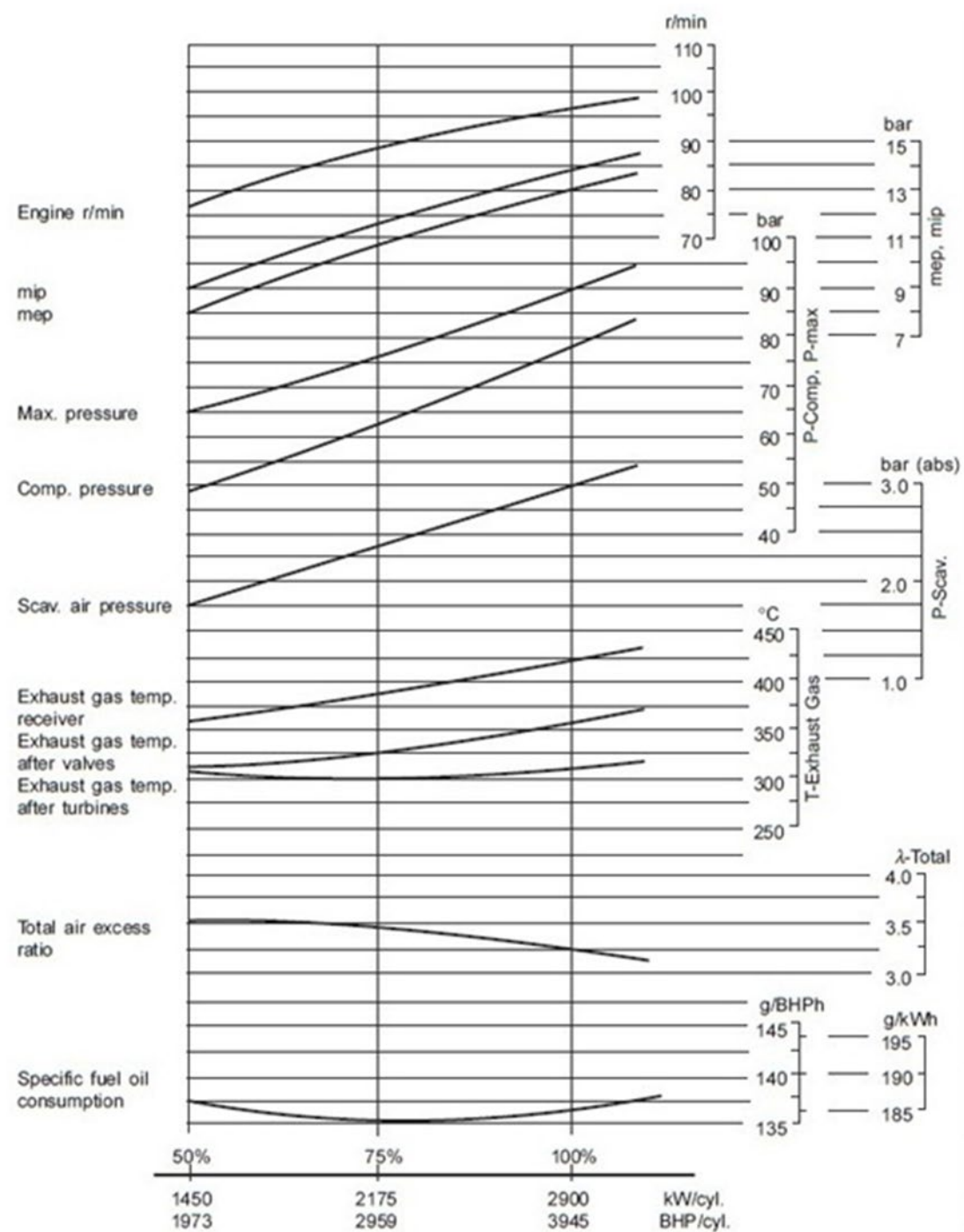
Diesel Engine Acceptance Test Record

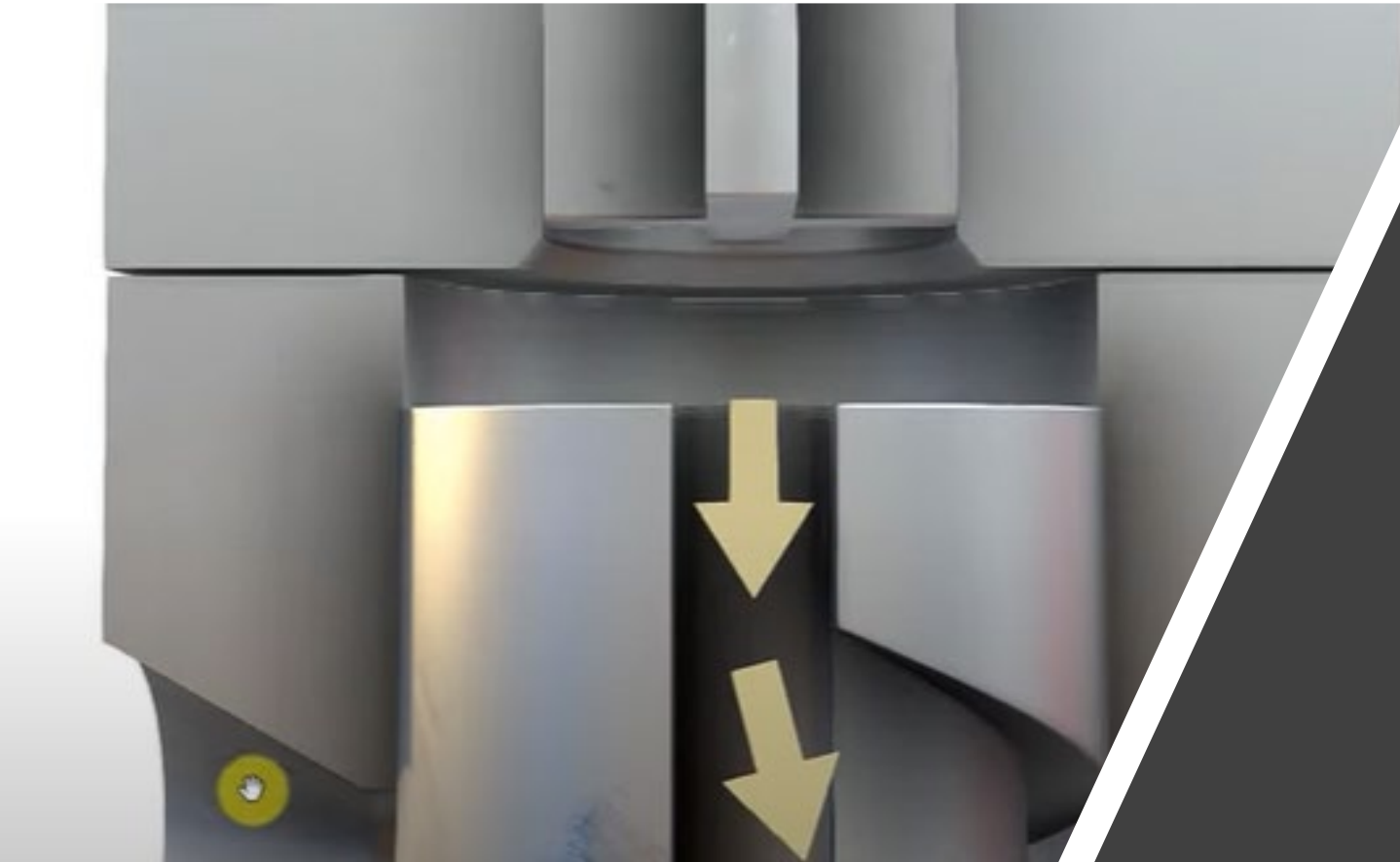
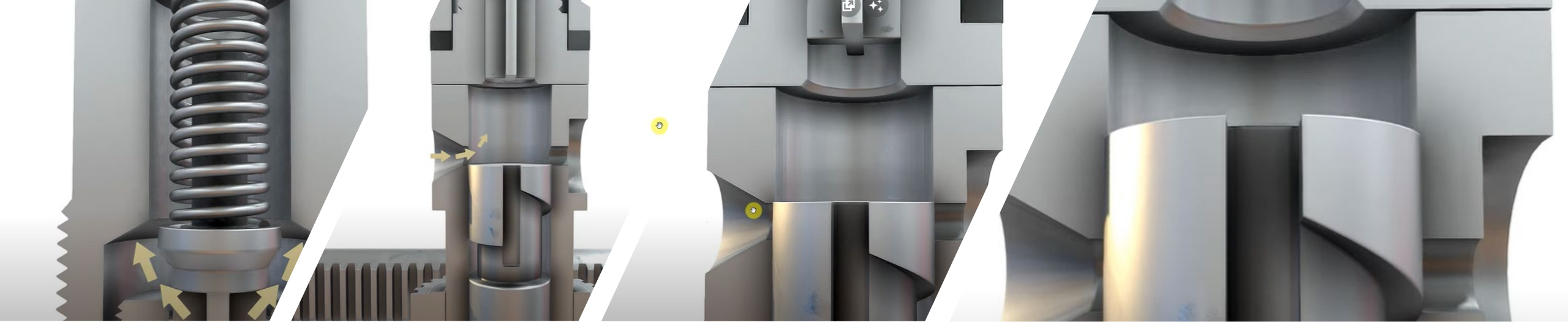
Engine Type: 8 M 20				Order No.: 251678				Engine No.: 31035																											
Atmospheric conditions during test run				Ambient temperature: 26,0 °C				Atmospheric pressure: 0,986 bar				Relative humidity: 65 %				Altitude: 10 m																			
We used for the test run				Luboil type: SAE 40				Heat value according to Fuel oil analysis: 42290 kJ/kg				Density: 0,884 kg/l				Test bed: 50 / 8				Hydraulic brake: 12 U 2 N 86															
Engine data:				Four-stroke / direct injection / port engine / counter-clockwise rotation (viewed from the drive end)												Firing order: 1-3-5-7-8-6-4-2																			
Rated power: 1520 kW				Rated speed: 1000 1/min				Bore: 200 mm				Stroke: 300 mm																							
Charger type: KBB R 4 - 3				Number: 733				Max speed: 41500 1/min				t max: 650 °C				Specification: 2.448-00207				Test: 3438 LEI 05															
Fuel injection pump: Woodward MWA 205615				Plunger diameter: 19,0 mm				Idle stroke 'X': 4,0 mm				Emissions data																							
Fuel injector: Mak A 3 / 5				Opening pressure: 500 bar								Application : Constant Speed Main Propulsion																							
Generator Manufacturer:				Type:								Cycle : E2				Test result of parent engineNOx : 9,9 g/kWh																			
Flywheel diameter: 1005 mm				1° : 8,77 mm				Commencement of injection pump Cylinder 1 : 6,7° crank angle before T.D.C																											
Settings:												Cyl 1		Cyl 2		Cyl 3		Cyl 4		Cyl 5		Cyl 6		Cyl 7		Cyl 8									
Distance from liner top edge to piston top edge in T.D.C. [mm]												12,5		12,5		12,5		12,5		12,5		12,5		12,5		12,4									
Fuel injection pump thickness of the Steel Plate [mm]												2,7		2,4		2,5		2,2		2,5		2,3		2,2		2,2									
Fuel injection pump rack position when control handle on 'stop' [mm]												0,0																							
Fuel injection pump rack position [mm]												at rated power		26,0																					
At control panel blocked at 26,0 mm												at no load		11,0																					
Governor speed setting n = 1000 1/min												Maximum speed no load n = 1000 1/min												Minimum speed no load n = 597 1/min											
Run time	Fuel	Output				Mean eff. press pme bar	Fuel consumption					Lubricating oil			Cooling water			Charge air cooler °C				Exhaust gas temperature °C								Turbocharger					
		Speed	Pe	Load			Quant	Run time	Consumption			Press.	Temp.	Out	Press.	Temp.	Out	Water	Air	Cylinder				aft. Turb.	Charge air pr. cool.	aft. cool.	Diff. press	Speed							
min.	mm	rpm	kW	kN	%	kg	s	g/kWh	ISO	kg/h	bar	°C	°C	bar	°C	°C	In	Out	In	Out	1	2	3	4	5	6	7	8	bar	bar	mbar	rpm			
60	26,0	1000	1520	15,2	100	24,19	4,0	48,4	195,7	190,9	297,5	4,7	60	66	4,0	59	64	46	56	214	50	392	382	385	383	404	392	387	376	316	3,150	3,125	25	39000	
30	28,0	1000	1672	16,7	110	26,61	4,0	43,7	197,0	192,2	329,4	4,7	62	70	4,0	59	64	46	58	232	51	420	407	411	408	431	416	412	405	324	3,510	3,480	30	40800	
30	23,0	1000	1292	12,9	85	20,56	4,0	57,2	194,9	190,1	251,8	4,7	64	72	3,8	61	64	46	58	192	50	366	363	365	359	382	368	368	359	319	2,580	2,560	20	36500	
30	21,5	1000	1140	11,4	75	18,14	4,0	64,5	195,8	191,0	223,2	4,7	64	72	3,9	62	65	46	55	177	50	354	354	355	354	371	359	357	354	323	2,250	2,235	15	35000	
30	17,0	1000	760	7,6	50	12,10	4,0	90,9	208,4	203,6	158,4	4,7	63	70	3,9	63	65	44	52	129	48	337	343	342	338	359	343	348	338	354	1,250	1,240	10	28300	
30	13,0	1000	380	3,8	25	6,05	4,0	160,7	235,8	230,7	89,6	4,7	65	69	3,9	62	63	41	42	72	46	341	355	344	335	360	348	337	334	356	0,410	0,400	10	18500	
20 is basic value for crankweb deflection 1/100 mm						Measured on test bed and coupled to hydraulic brake.												Notes:												Acceptance					
						Measured with overhung flywheel.												Specific fuel consumption including 1 lubricating oil pump and 1 cooling water pump .												BV					
						Follow MaK instructions for installation.												Engine elastically mounted .												BV 0626 HBR 06					
						Exhaust gas back pressure at full load: 0,011 bar																								Mechanic					
						Cylinder 1																		17.02.2006 / Kahmann											
B.D.C.		Exhaust side cold		warm		T.D.C.		cold		warm		Camshaft side cold		warm								Engineer													
20		21,0		19,5		20,0		17,5		20,0		17,5								27.02.2006 / M. Mohr															
																		1) No. 1 cylinder on flywheel end of engine												Approved					
																		2) Cooling water temperature: Inlet oil cooler: 38 °C												<i>[Signature]</i> 27.02.06					
Caterpillar Confidential: green						DICARE f. Windows / Version 2.0 / 20.01.2006																													

SHOP TRAIL DATA

stx		4. Summary Data of Shop Trial						Checked by: S. H. Lee	
Engine No. : SB6S50-7161		Owner		COSCO GROUP		Output		9,480 kW	
Hull No. : ZS07035		Shipyard		COSCO ZHOUSHAN		Test Date		2008. 08. 12.	
	Unit	1	2	3	4	5	6	7	
Load	%	25	50	75	90	100	110		
Engine speed	rpm	80.0	100.8	115.4	122.6	127.0	131.1		
Engine output	kW	2,370	4,740	7,110	8,532	9,480	10,428		
Room temperature	°C	29.0	29.0	28.0	30.0	31.0	31.0		
Room humidity	%	82.0	82.0	80.0	78.0	74.0	70.0		
Barometer pressure	mbar	1013	1013	1013	1013	1011	1010		
Fuel oil temperature	°C	36.0	38.0	38.0	38.0	38.0	38.0		
Fuel oil consumption	kg/h	444.6	843.0	1238.4	1491.0	1670.4	1863.0		
Fuel oil consumption	g/kW/h	187.59	177.85	174.18	174.75	176.20	178.65		
Fuel oil consumption (ISO.)	g/kW/h	186.16	176.38	172.78	173.17	174.67	177.10		
Max.Combustion pressure	MPa	7.47	10.08	13.27	14.98	14.98	14.95		
Compression pressure	MPa	4.50	7.02	9.93	11.52	12.38	13.65		
Mean effective pressure	MPa	0.75	1.20	1.57	1.77	1.90	2.03		
Fuel injection pump index	mm	39.0	53.3	66.2	75.0	80.0	86.0		
Exh.temp cylinder outlet	°C	282.5	298.3	298.3	315.0	333.3	360.8		
Exh.temp T/C inlet	°C	315.0	350.0	360.0	390.0	410.0	450.0		
Exh.temp T/C outlet	°C	275.0	270.0	240.0	240.0	250.0	265.0		
Turbocharger speed (X 1000)	rpm	5.8	10.1	12.7	13.7	14.4	15.1		
Scav.air temperature	°C	37.0	32.0	35.0	38.0	38.0	39.0		
Scav.air pressure	MPa	0.030	0.100	0.180	0.220	0.250	0.290		
Turbocharger gas inlet press	MPa	0.020	0.080	0.160	0.200	0.230	0.270		

Performans diyagramı

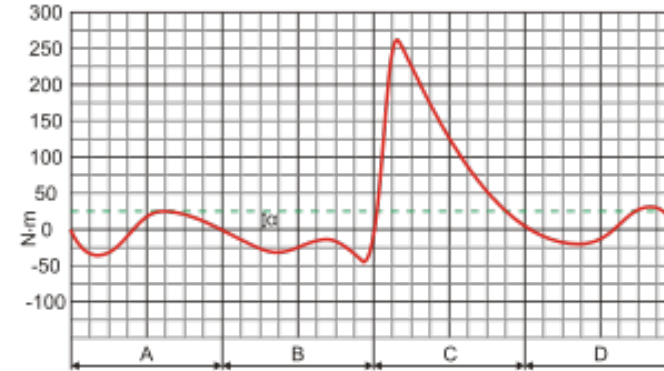
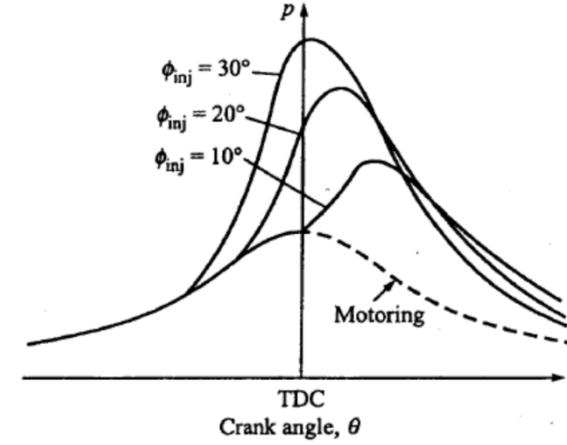
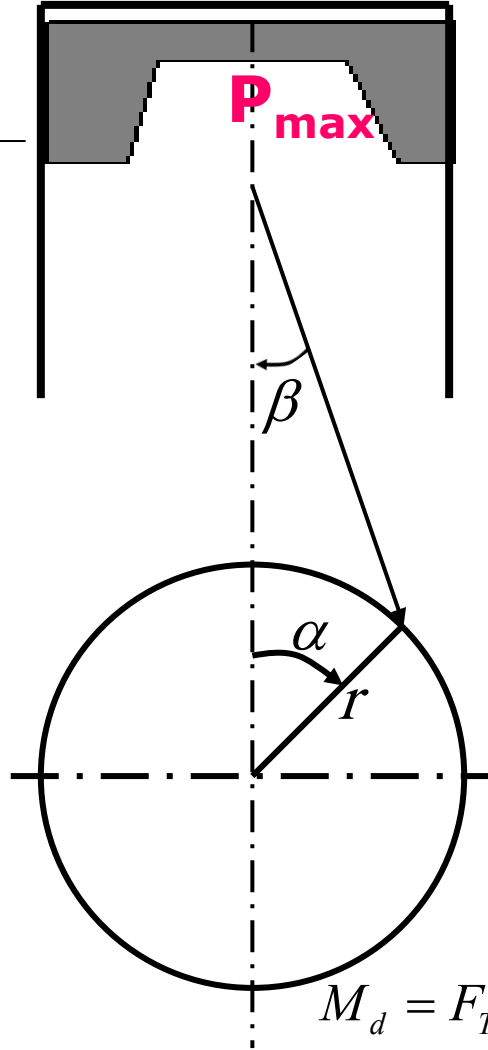
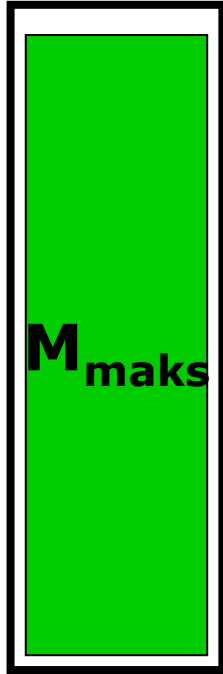




Pompa Basma
başlangıcı ve Avans
İlişkisi

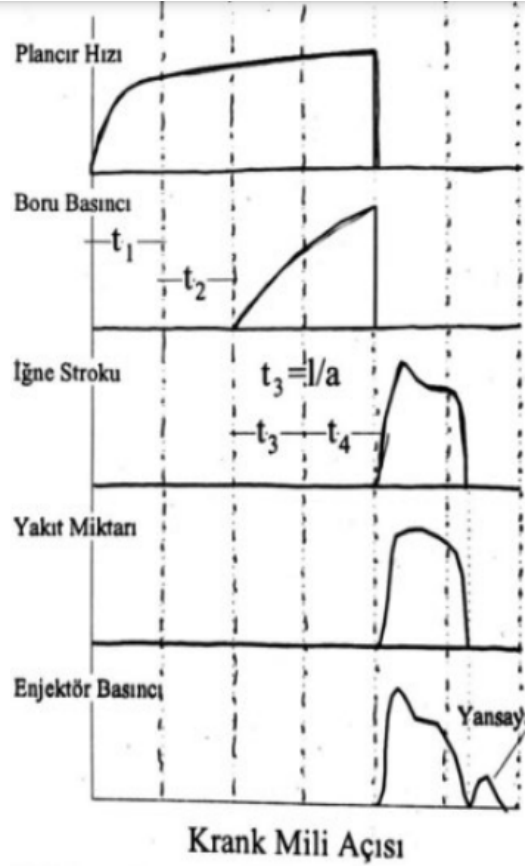
GÜÇ ve MOMENT ne ZAMAN MAKSİMUM

15-20 °KMA



$$M_d = F_T \cdot r = F_{yanma} \sin \alpha \left\{ 1 + \frac{R \cdot \cos \alpha}{[1 - R^2 \sin^2 \alpha]^{1/2}} \right\}$$

Basma Başlangıcı-Püskürtme Gecikmesi-Püskürtme Gecikmesi



Şekil 1-Püskürtme gecikmesinin kademeleri

Püskürtme Kanunu: Püskürtülen yakıt miktarının krank mili açısına (veya zamana) göre değişimine denir. Püskürtme kanunu yanmanın gidişini etkiler. Buna göre püskürtme zamanlaması motordan en yüksek verim alınacak, ancak motorda mekanik ve ısı zorlanma olmayacak şekilde ayarlanması gerekir. Bu, yanma sonucu oluşan maksimum basıncın ÜÖN dan 10-15 °KMA oluşması ile sağlanmaktadır.

Püskürtme Gecikmesi(t): Yakıt pompa plencerinin harekete geçmesiyle enjektör iğnesinin açılması arasında geçen zamandır. *Püskürtme gecikmesi* olarak adlandırılan bu süre çeşitli kademelerden oluşur. Bu kademeler,

$$t = t_1 + t_2 + t_3 + t_4$$

t₁:Pompalama gecikmesi-pistonun hareket başlaması ile emme kanalının kapanması arasındaki zaman.

t₂:Sevk gecikmesi-Portun kapanmasıyla basın dalgasının yola çıkması arasındaki zaman.

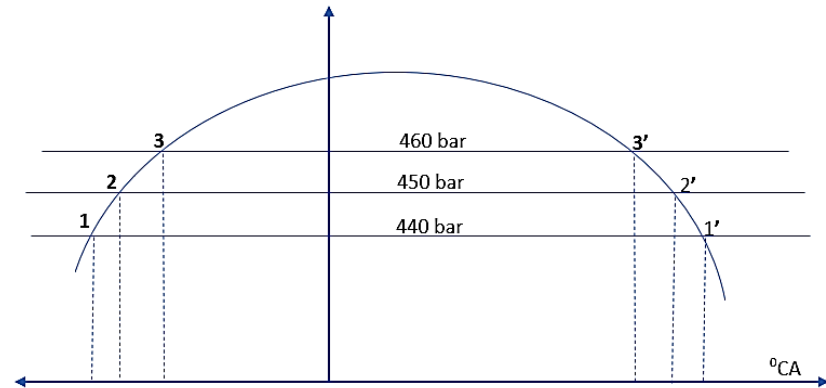
t₃:Yol gecikmesi-Dalganın harekete geçmesiyle enjektöre ulaşması arasındaki zaman

t₄:Açılma gecikmesi-Dalganın enjektöre ulaşmasıyla enjektörün açılması arasında geçen zaman

- ❖ Basma başlangıcı/start of Delivery
- ❖ Püskürtme Gecikmesi
- ❖ Tutuşma gecikmesi

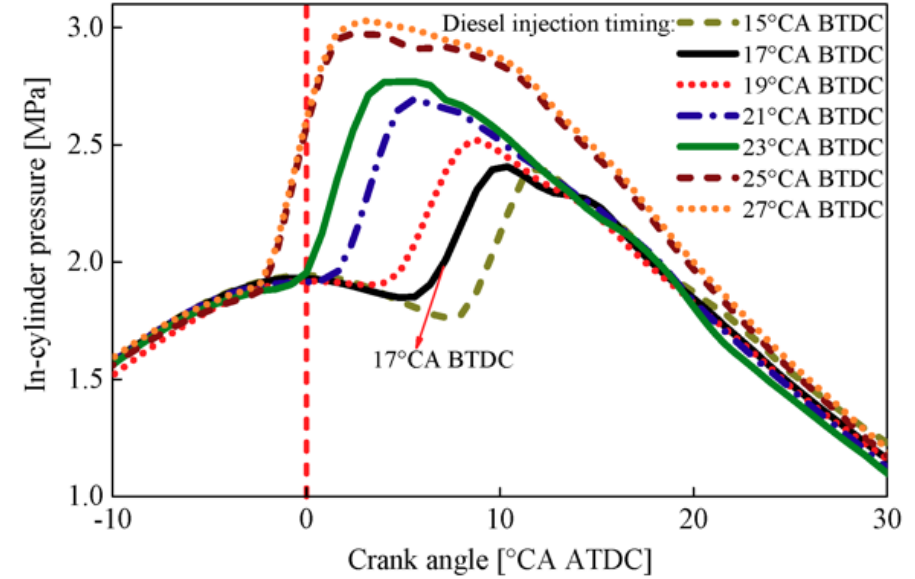
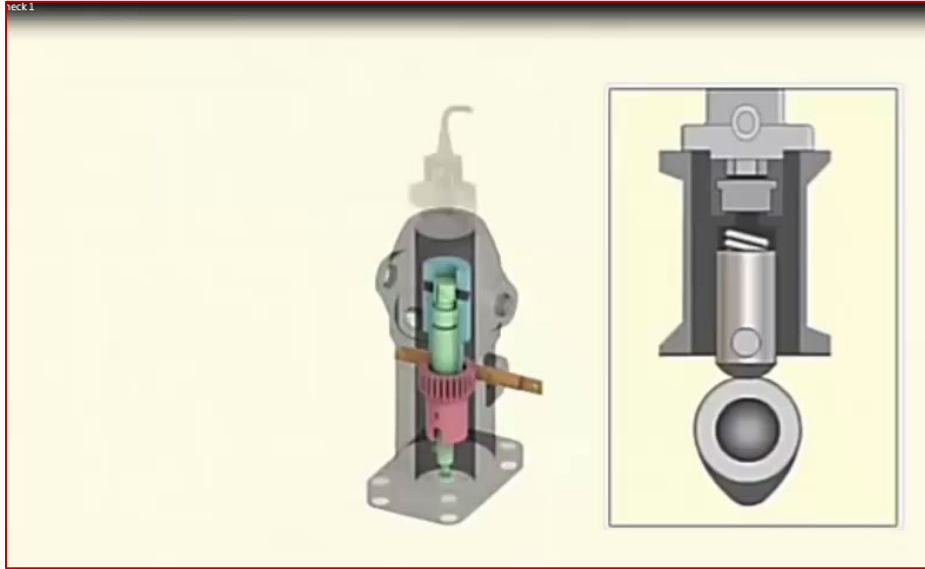
- Yakıt pompa delivery valf değişirse
- Enjektör yayı farklı olursa
- Pompa basma basıncı değişirse
- Enjektör açma basıncı değişirse

Püskürtme Avansı değişir.



Enjektör Açma Basıncı Değişiminin Avansa etkisi

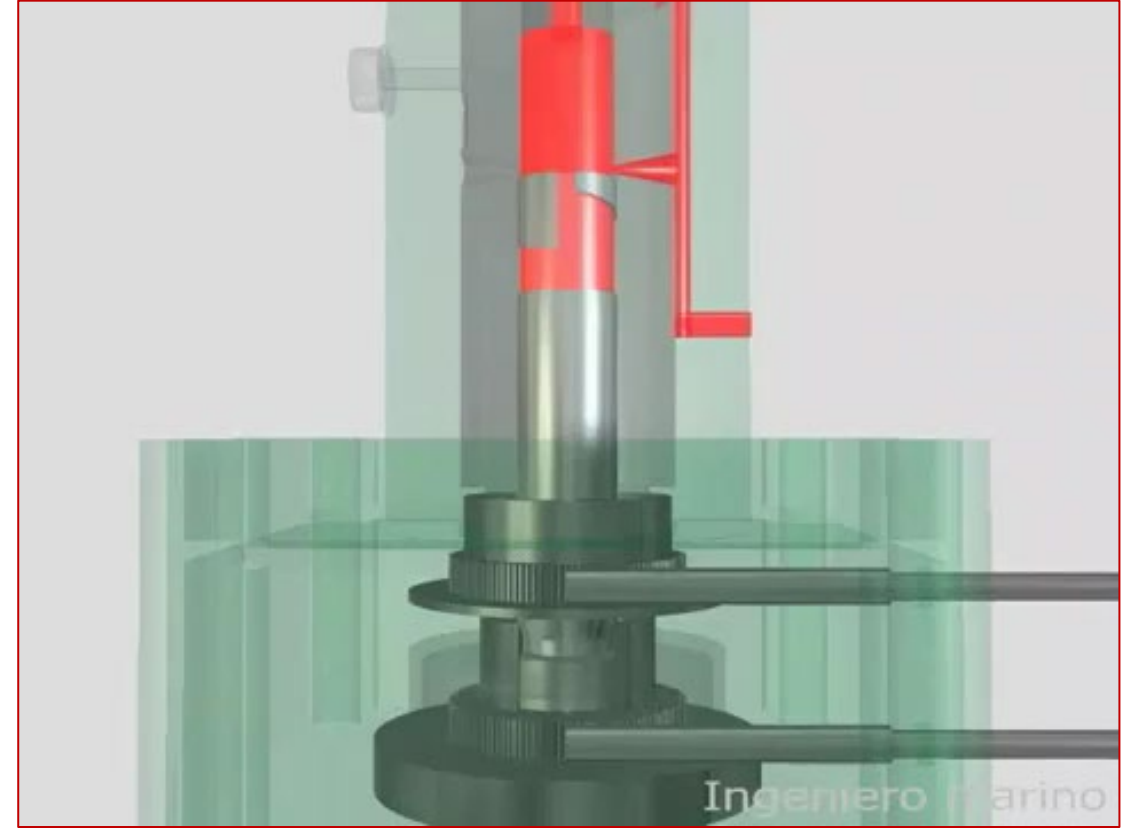
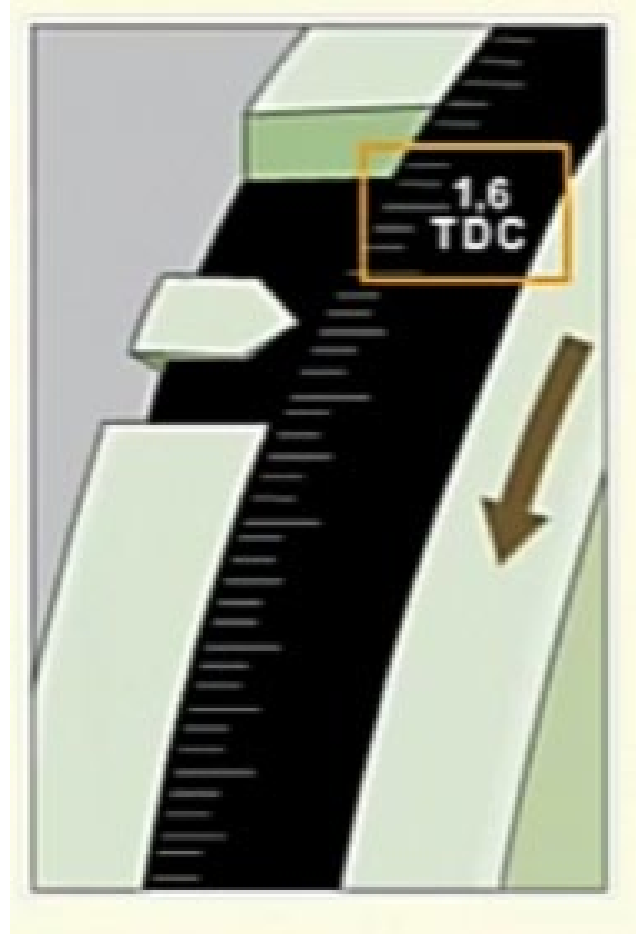
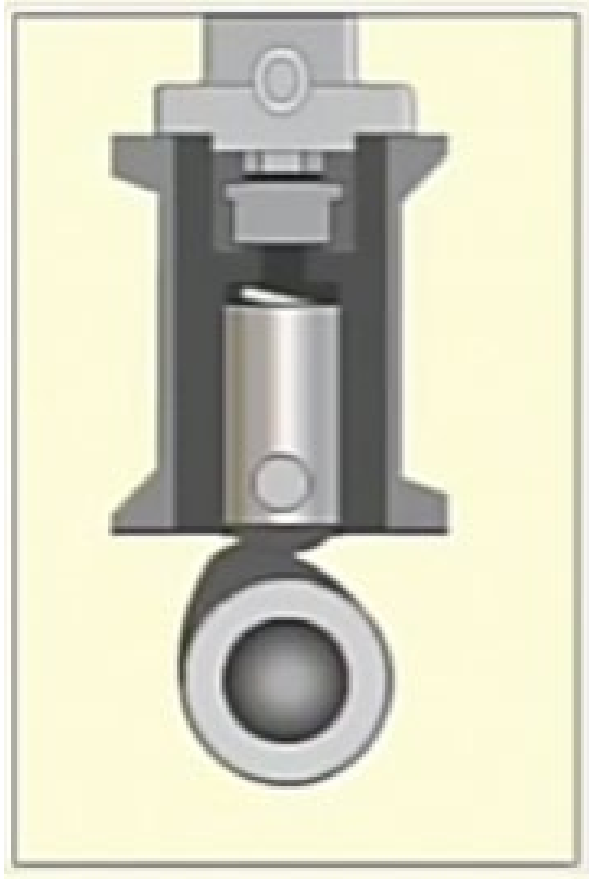
Yakıt Pompası Püskürtme Başlangıcı (Statik Avans)



Püskürtme avansı, yakıtın yanma odasına püskürtüldüğü zamanı ifade eder. Yakıt özellikleri ve motor karakteristiğine bağlı olarak silindirde maksimum basıncın maksimum torku sağlayacak şekilde ayarlanır.

VIT-Püskürtme Avansını Ayarlama...

Püskürme Avansı VIT koluyla otomatik yapılmaktadır. Barıl hareket ettirilerek basma başlangıç açısı değiştirilmektedir.

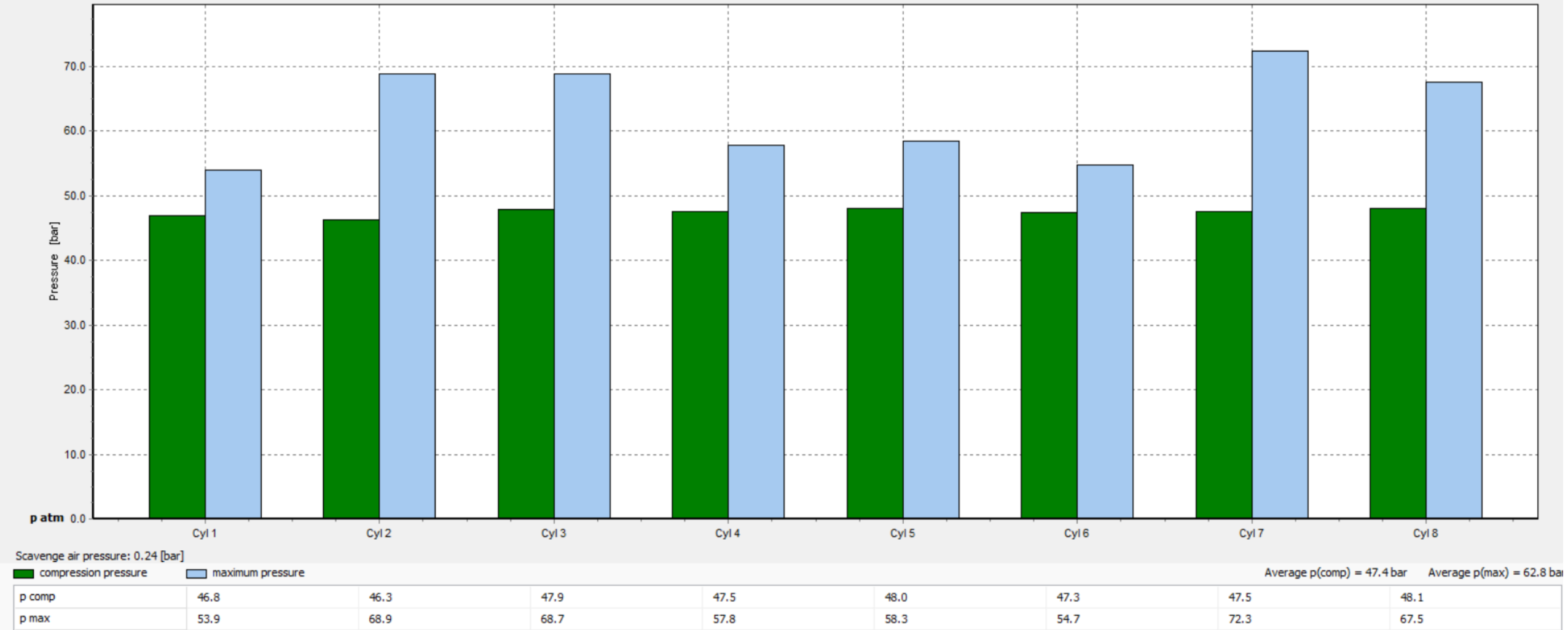




MAK 8M20

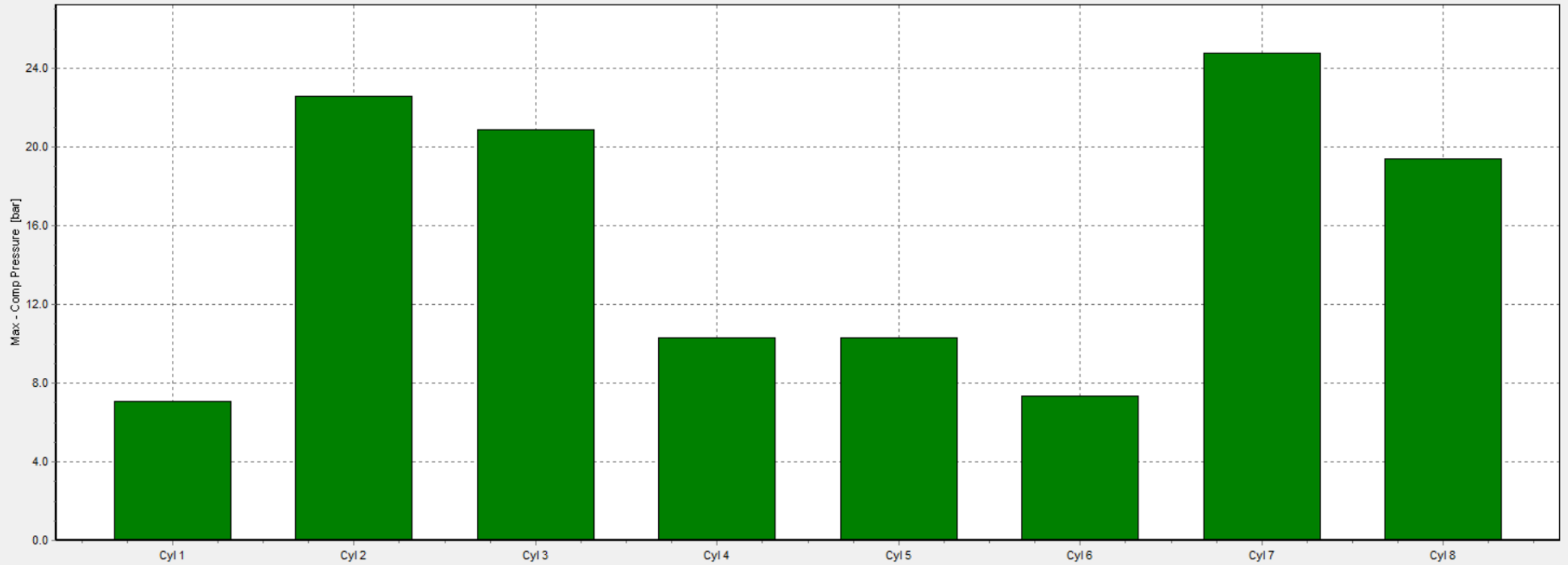
MAK 8M20

P_{\max} ve P_{com} ölçümüne göre ölçüm yaptığınız ana makine hakkındaki yorumunuz ve çözüm öneriniz nedir?



Pmax-Pcom

Pmax-Pcomp arasındaki farklılığın sebepleri neler olabilir?



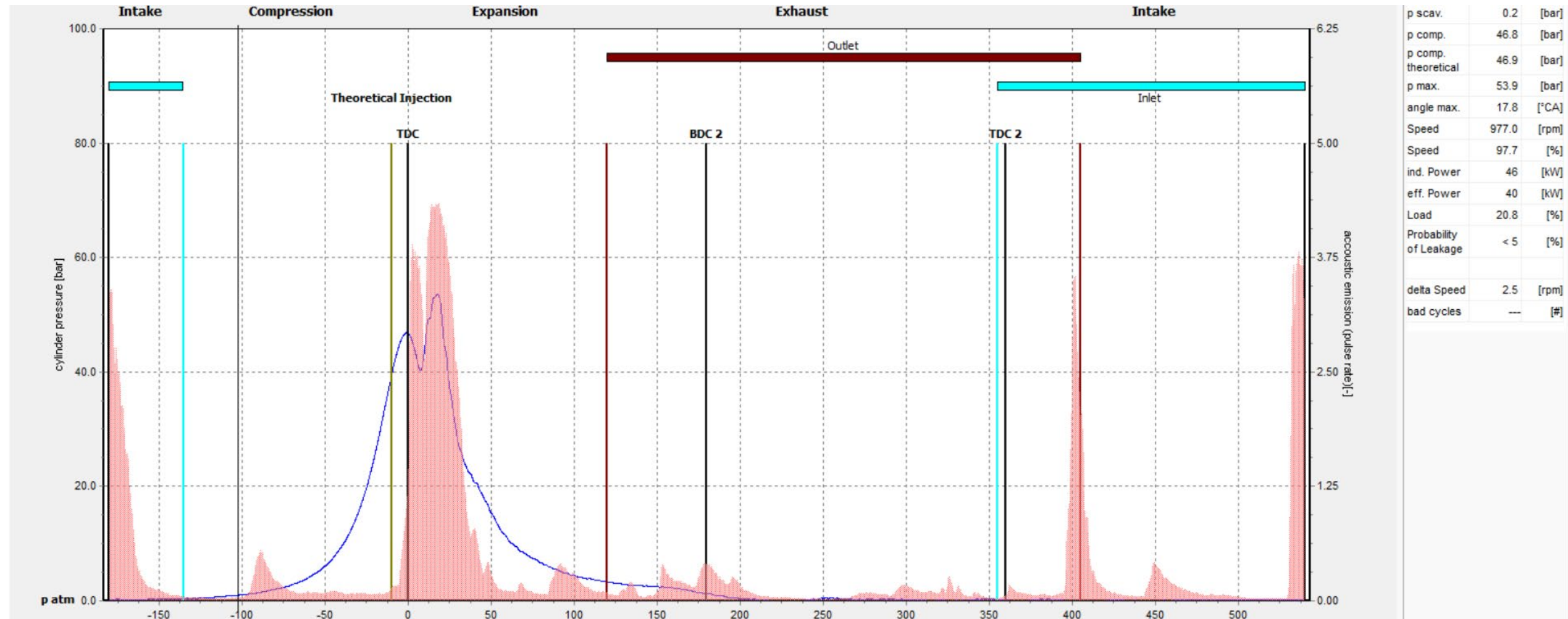
Scavenge air pressure: 0.24 [bar]

Max - Comp Pressure

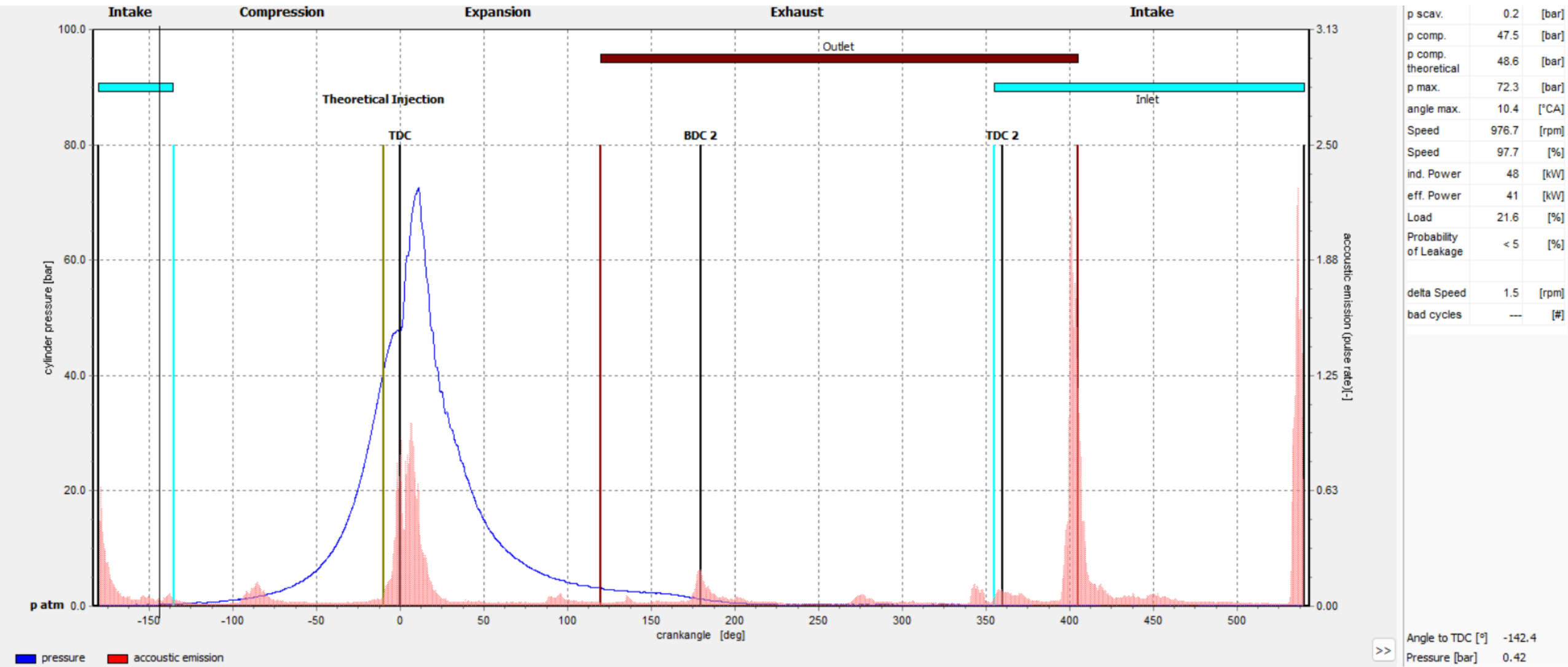
Average Max - Comp Pressure = 15.32 bar

Pmax - Pcomp	7.06	22.59	20.85	10.27	10.32	7.33	24.75	19.39
--------------	------	-------	-------	-------	-------	------	-------	-------

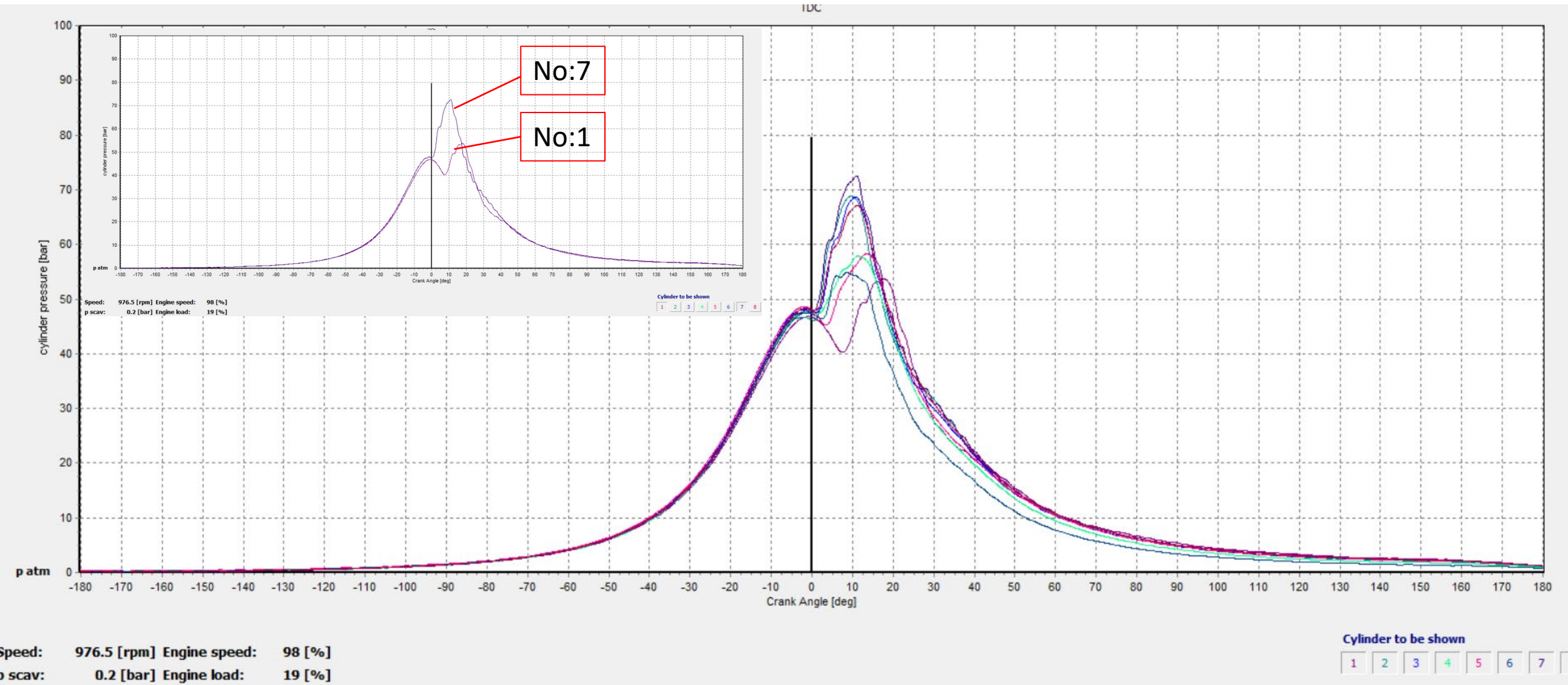
No:1 Silindire ait basınç gidişi ve püskürtme avansı hakkındaki görüşleriniz nedir?



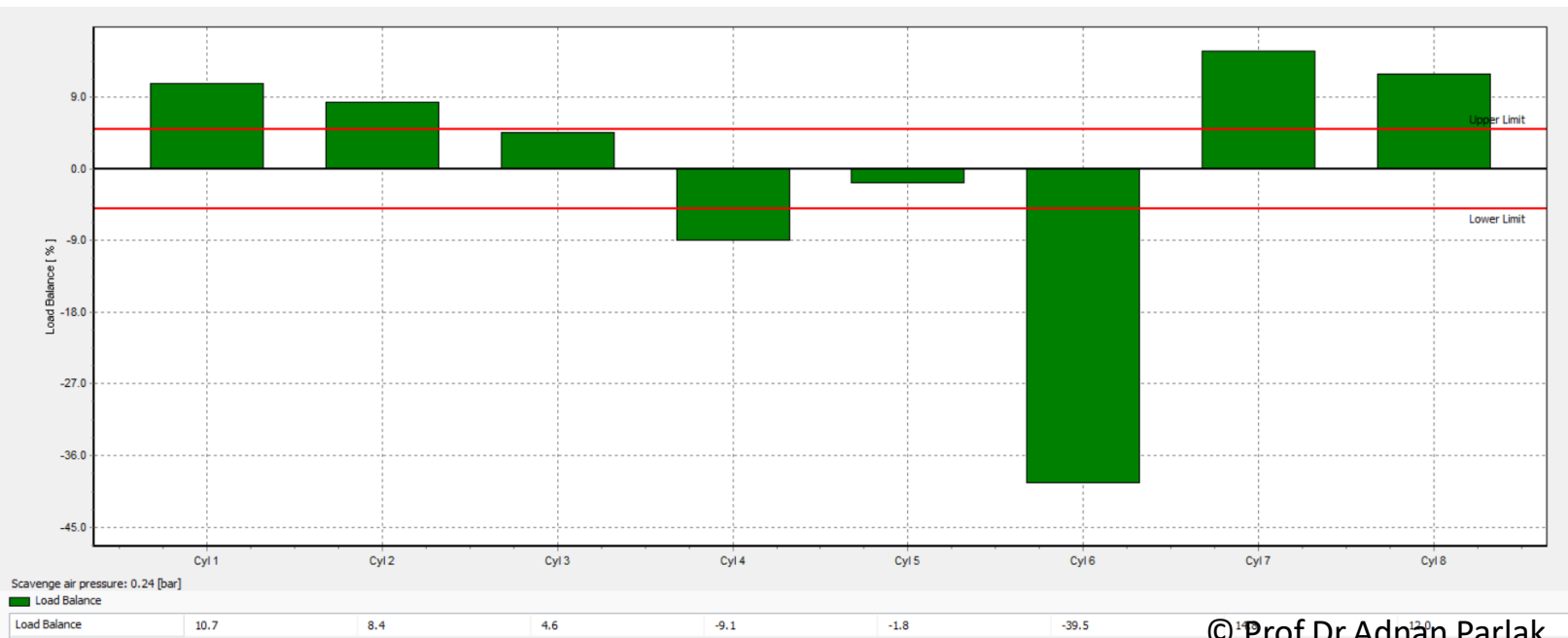
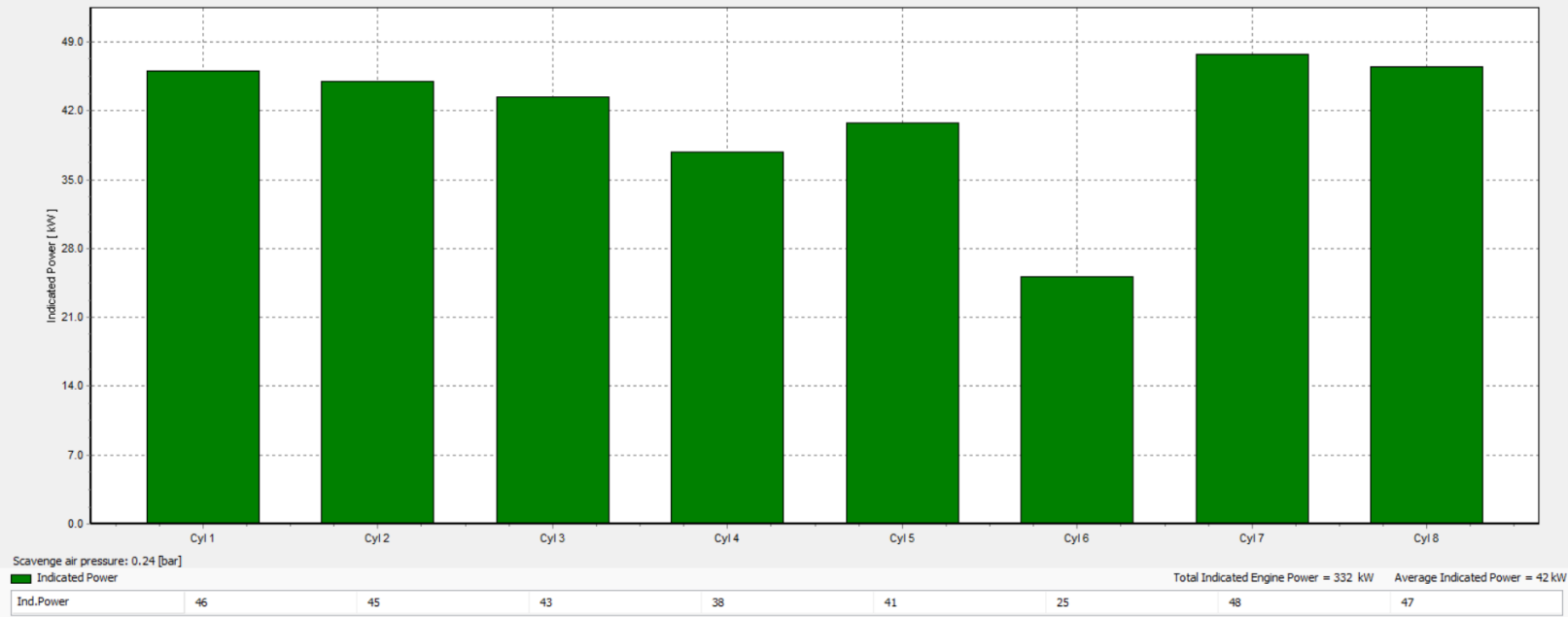
No:7 Silindire ait basınç gidişi ve püskürtme avansı hakkındaki görüşleriniz nedir?



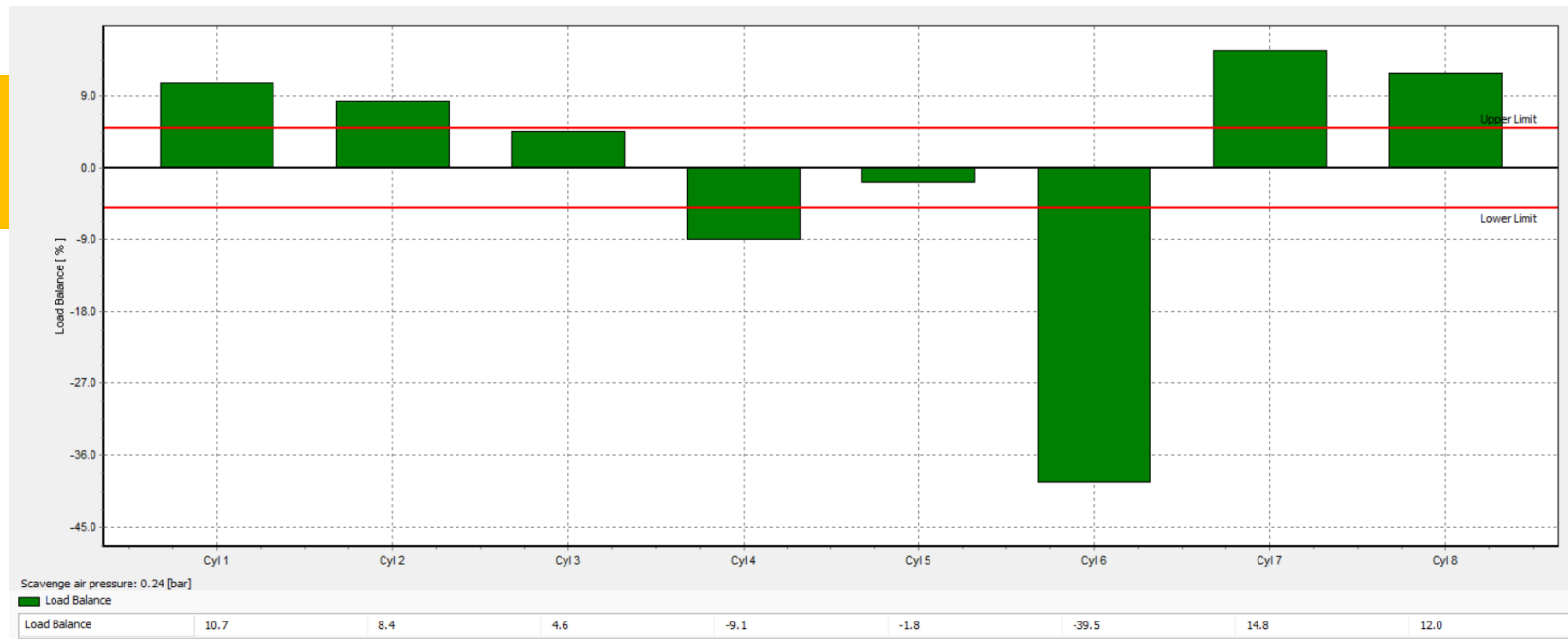
%20 Yükte basınç_krank açısı diyagramı



%20 yük Güç ve Yük Dağılımı



%20 yük Güç ve Yük Dağılımı

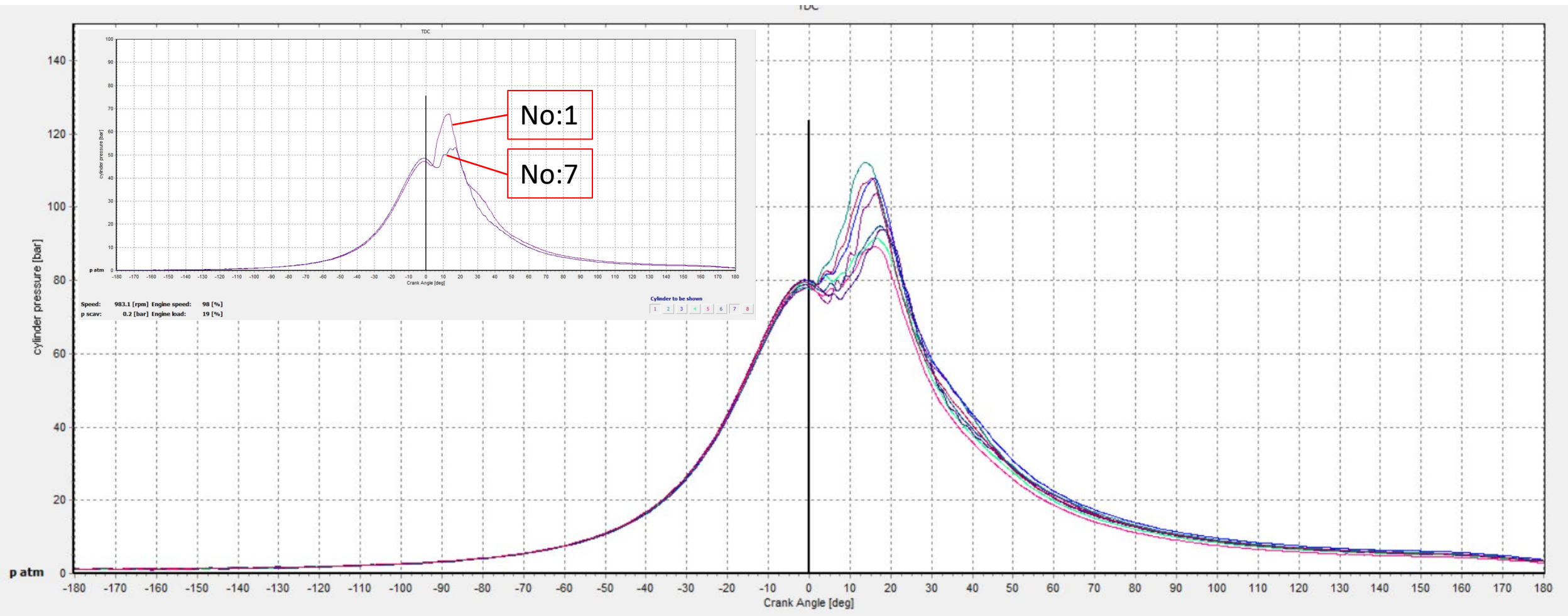


ind. power: 332.5 [kW] -> mech. efficiency: 0.861 -> eff. power: 286.2 [kW] -> electrical efficiency: 0.94 -> el. power: 269.0 [kW]

Scavenge air pressure: 0.24 [bar]

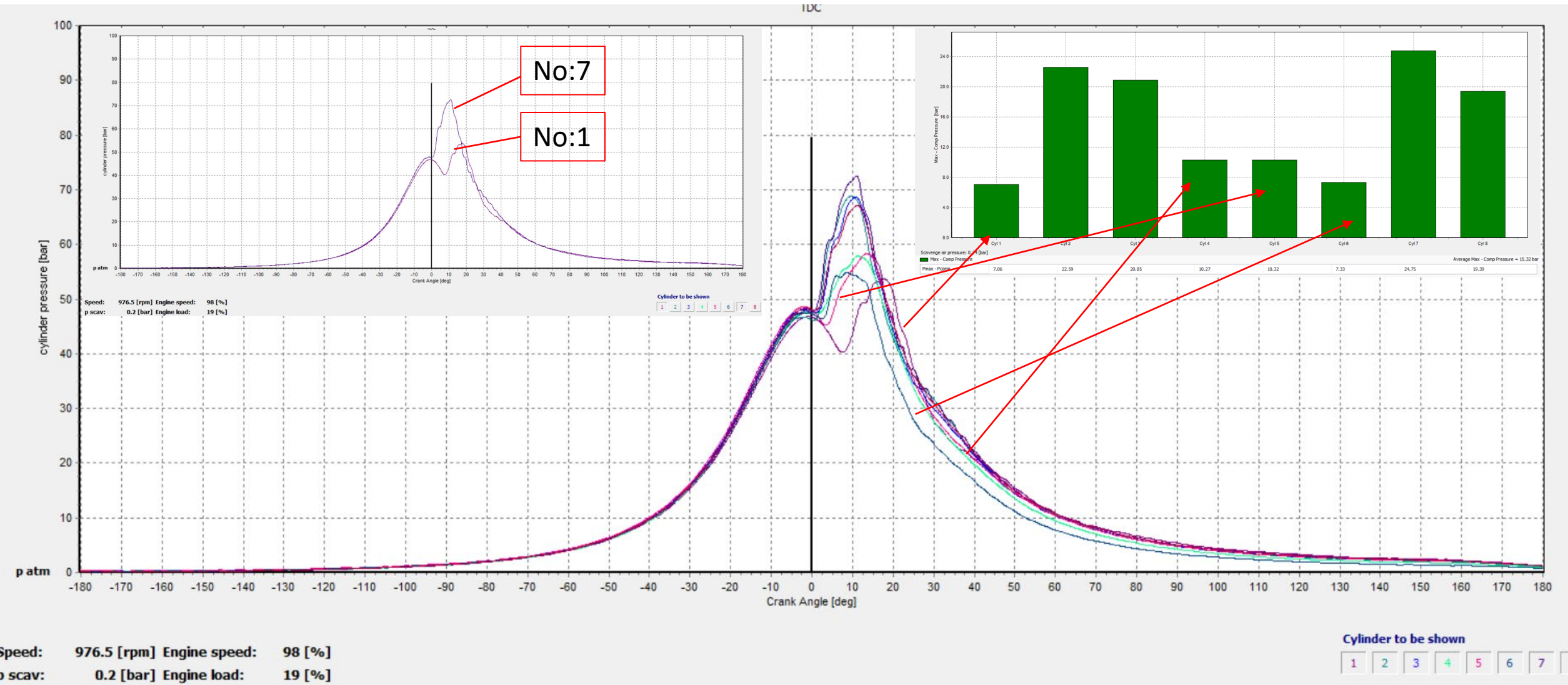
Value	Speed	Speed	p comp	p max	Angle p max	p comp/p scav	MIP	indicated Power	effective Power	Generator Power	Load	probable Leakage
Unit	[rpm]	[%]	[bar]	[bar]	[°Ca]	[-]	[bar]	[kW]	[kW]	[kW]	[%]	[%]
Cylinder 1	977.0	97.7	46.8	53.9	17.8	38.6	6.0	46.0	39.6	37.2	20.8	< 5
Cylinder 2	976.0	97.6	46.3	68.9	9.7	38.1	5.9	45.0	38.8	36.4	20.4	< 5
Cylinder 3	976.2	97.6	47.9	68.7	10.3	39.4	5.7	43.5	37.4	35.2	19.7	< 5
Cylinder 4	977.8	97.8	47.5	57.8	11.9	39.1	4.9	37.8	32.5	30.6	17.1	< 5
Cylinder 5	976.1	97.6	48.0	58.3	13.7	39.5	5.3	40.8	35.1	33.0	18.5	< 5
Cylinder 6	976.1	97.6	47.3	54.7	9.0	39.0	3.3	25.1	21.6	20.3	11.4	< 5
Cylinder 7	976.7	97.7	47.5	72.3	10.4	39.2	6.2	47.7	41.1	38.6	21.6	< 5
Cylinder 8	976.4	97.6	48.1	67.5	10.9	39.6	6.1	46.5	40.0	37.6	21.1	< 5
Total Value								332.5	286.2	269.0		
Average	976.5	97.7	47.4	62.8		39.1	5.4	41.6	35.8	33.6	18.8	
Maximum	977.8	97.8	48.1	72.3	17.8	39.6	6.2	47.7	41.1	38.6	21.6	
Minimum	976.0	97.6	46.3	53.9	9.0	38.1	3.3	25.1	21.6	20.3	11.4	
Difference	1.7	0.2	1.8	18.4		1.5	2.9	22.6	19.4	18.3	10.2	
Deviation [%]	0.2	0.2	3.8	29.4		3.7	54.3	54.3	54.3	54.3	54.3	

%44 Yükte basınç_krank açısı diyagramı

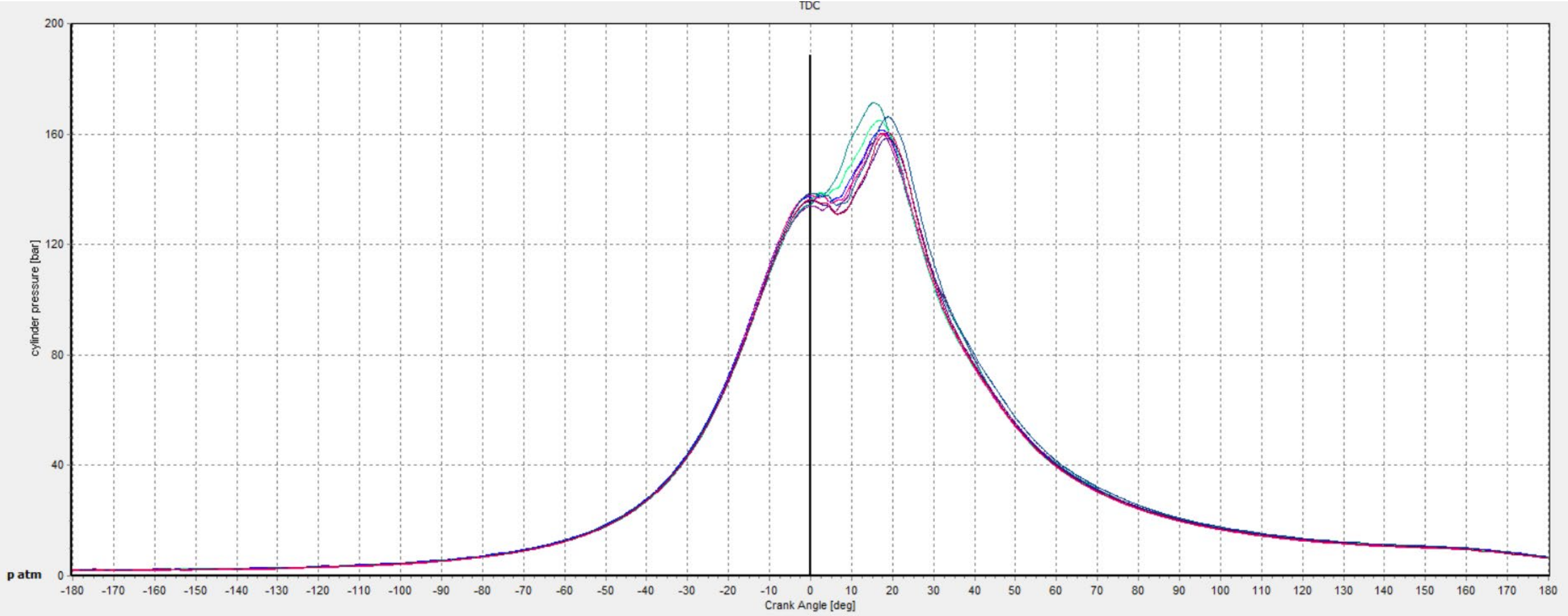


Cylinder to be shown
1 2 3 4 5 6 7 8

%20 Yükte basınç_krank açısı diyagramı



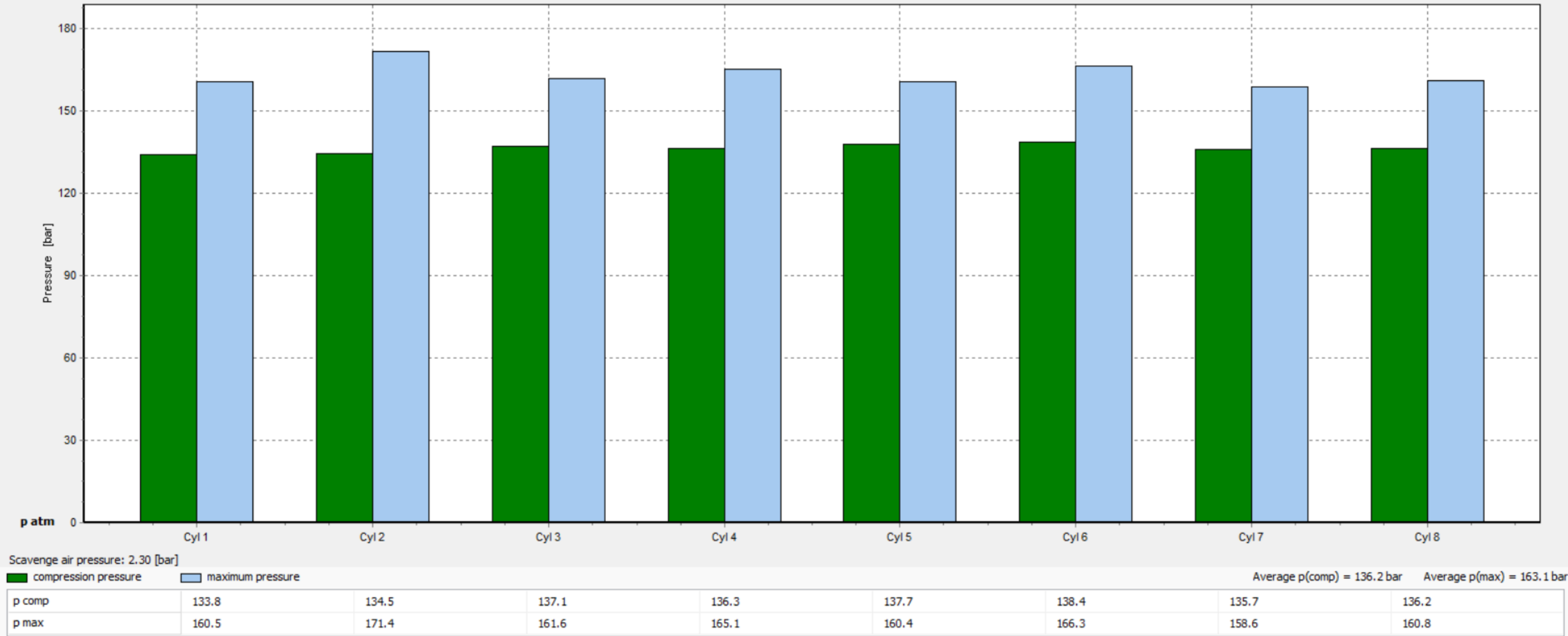
Pompa ve Enjektör deęiřimi ve ayarlama sonrası %93 yük



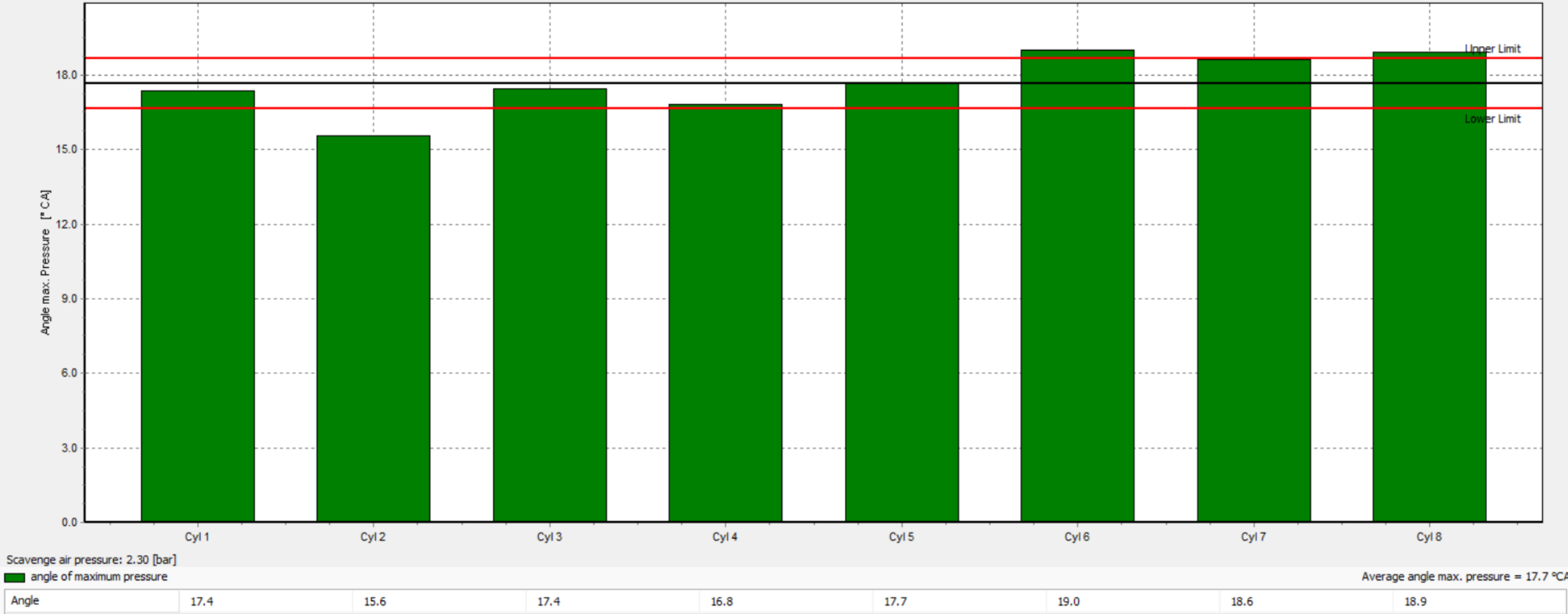
Speed: 971.7 [rpm] Engine speed: 97 [%]
p scav: 2.3 [bar] Engine load: 93 [%]

Cylinder to be shown
1 2 3 4 5 6 7 8

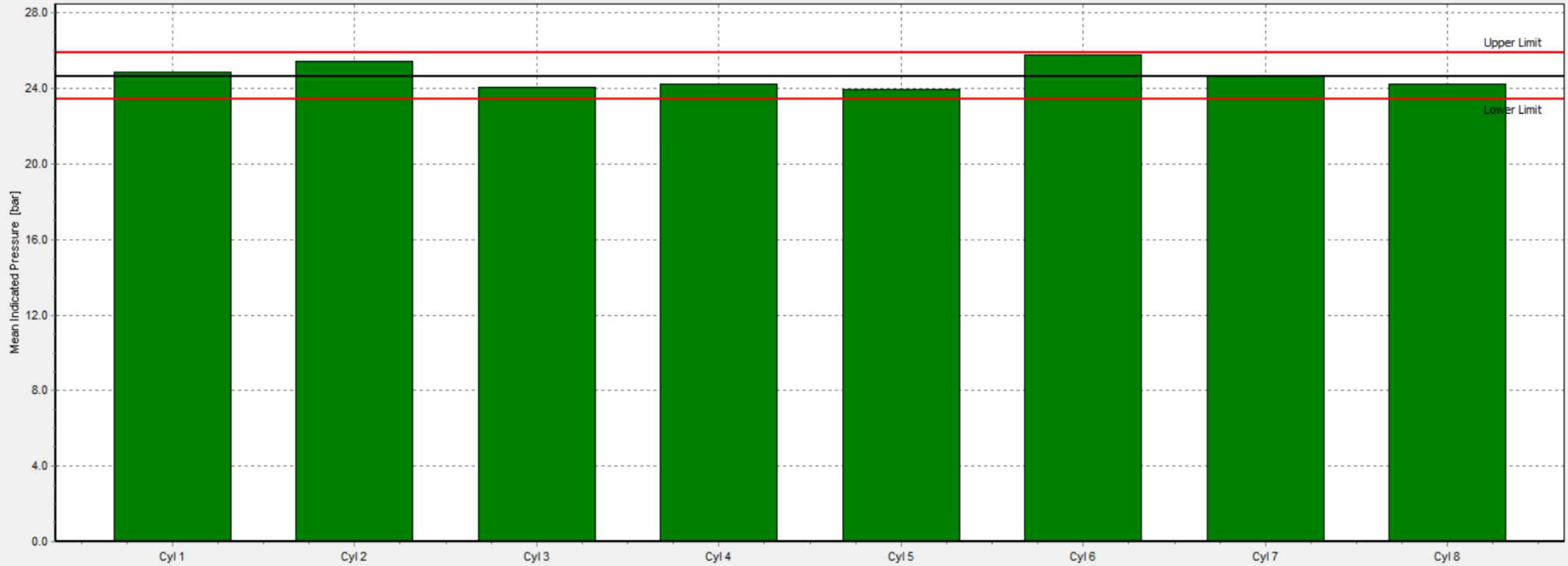
%93 yük Ayarlama sonrası Pmax ve Pcom değerleri



%93 yük Ayarlama sonrası *aPmax* değerleri



%93 yük Ayarlama sonrası **MIP** (*P_{mi}*) değerleri



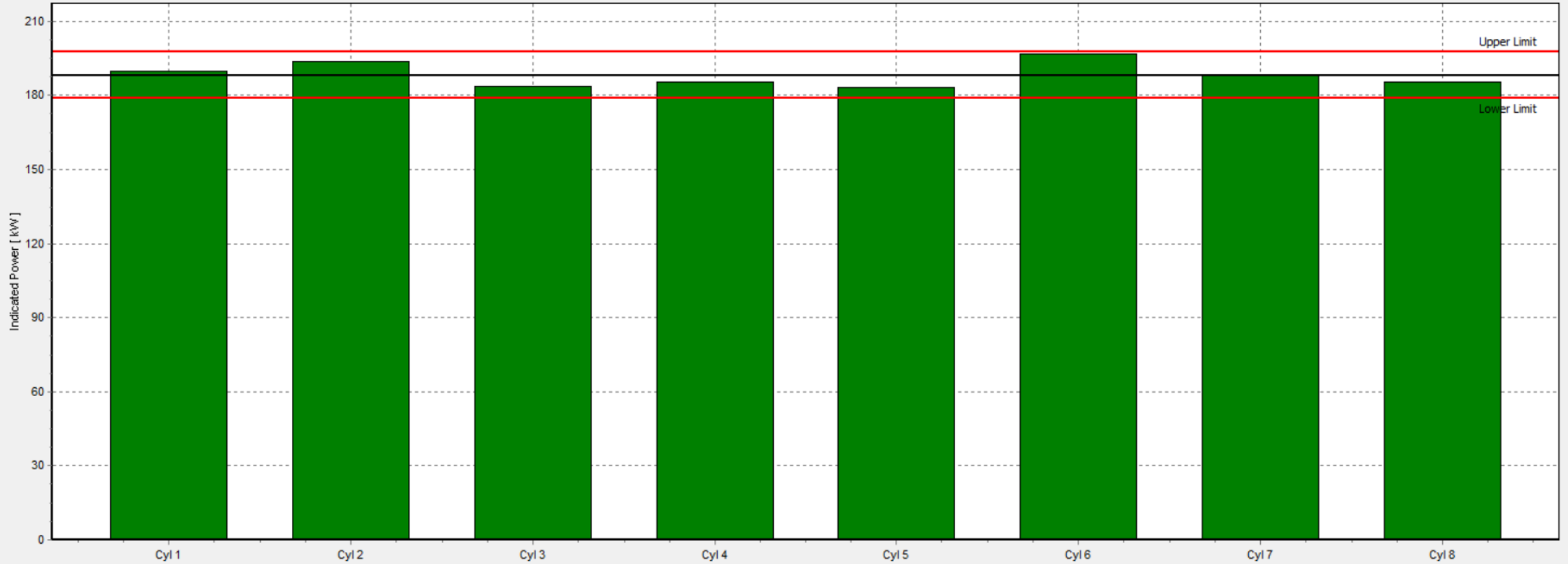
Scavenge air pressure: 2.30 [bar]

Mean Indicated Pressure (MIP)

Average Mean Indicated Pressure = 24.66 bar

MIP	24.86	25.45	24.07	24.24	23.98	25.81	24.63	24.25
-----	-------	-------	-------	-------	-------	-------	-------	-------

%93 yük Ayarlama sonrası Güç (P_e) değerleri



Scavenge air pressure: 2.30 [bar]

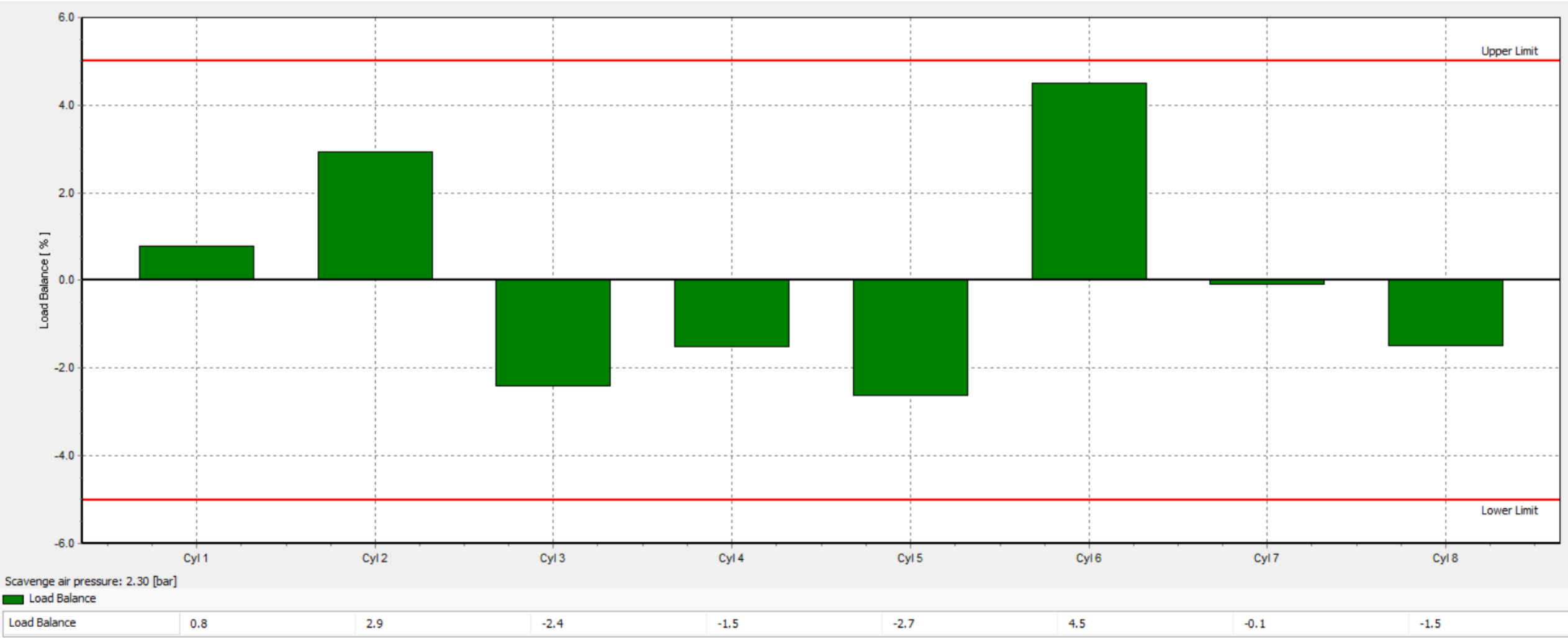
■ Indicated Power

Total Indicated Engine Power = 1505 kW

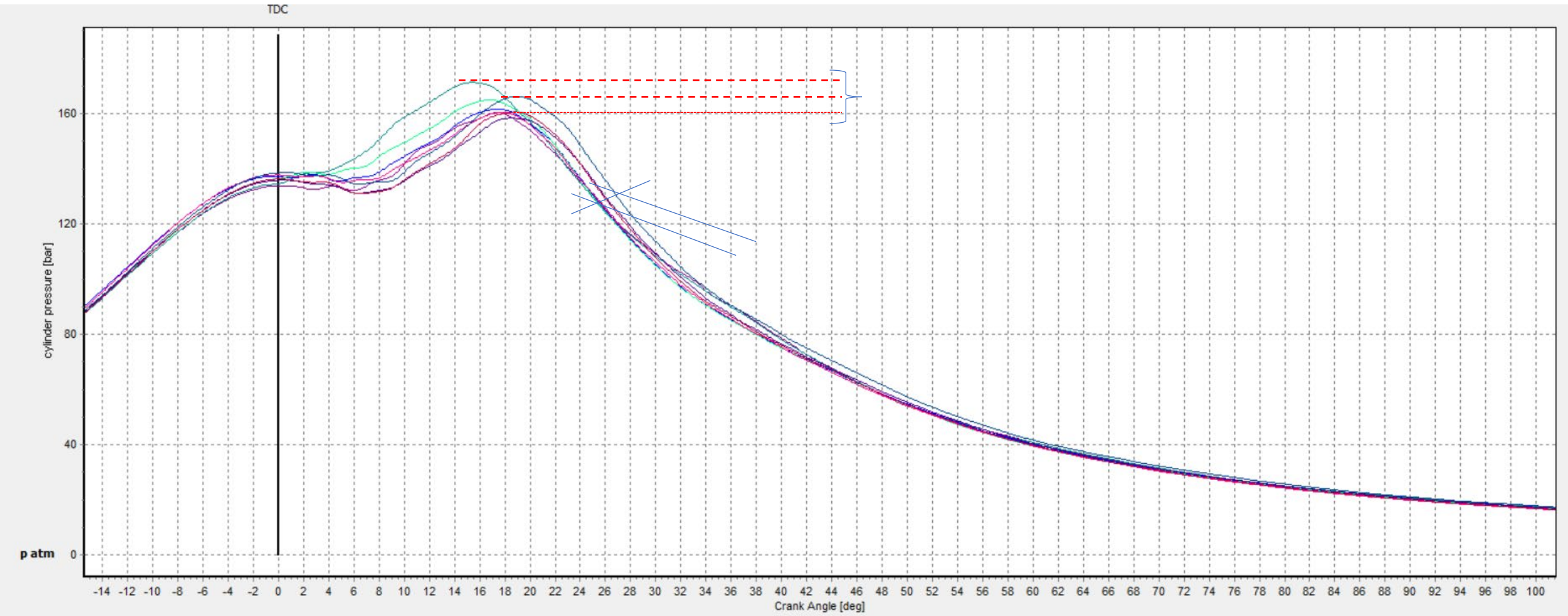
Average Indicated Power = 188 kW

Ind.Power	190	194	184	185	183	197	188	185
-----------	-----	-----	-----	-----	-----	-----	-----	-----

%93 yük Ayarlama sonrası Güç (P_e) değerleri



P_{max} değerlerinde farklılık var iken Yük dengesi $\pm 5\%$ lik dilim içerisinde düzgün dağılım sergiliyor?



Speed: 971.7 [rpm] Engine speed: 97 [%]
p scav: 2.3 [bar] Engine load: 93 [%]

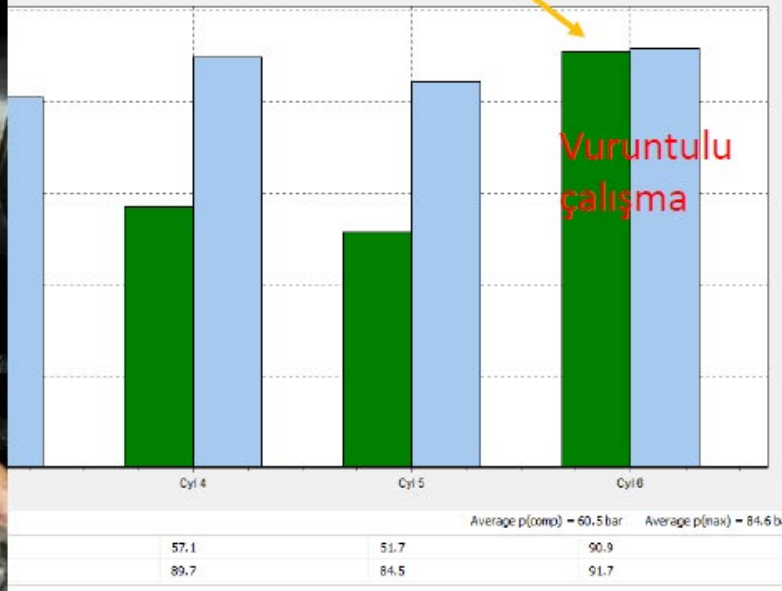
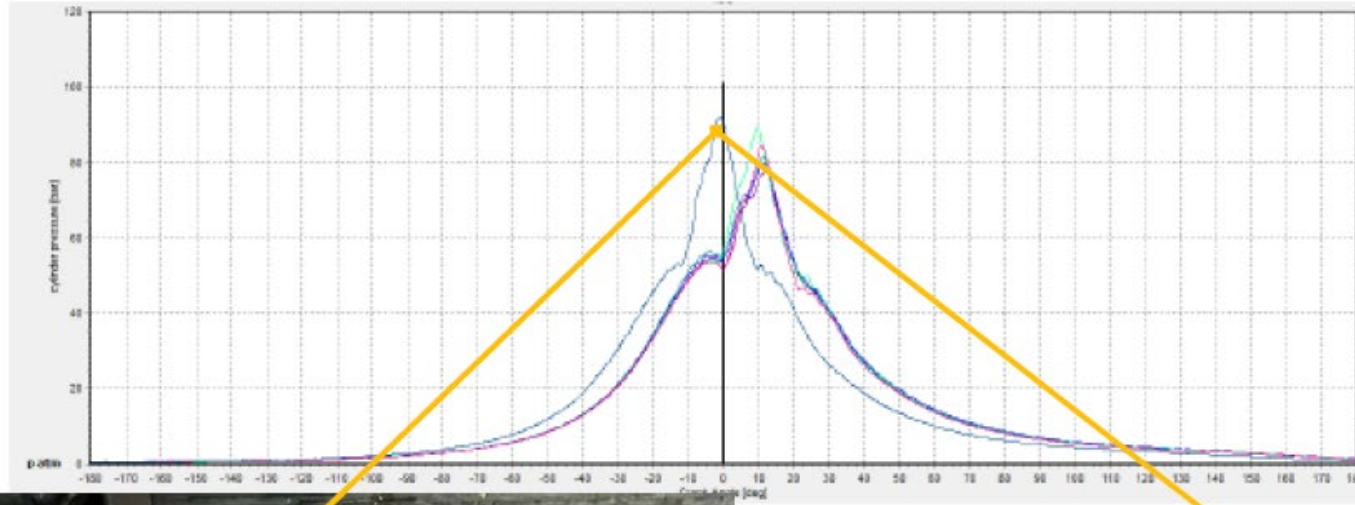
Cylinder to be shown

1 2 3 4 5 6 7 8

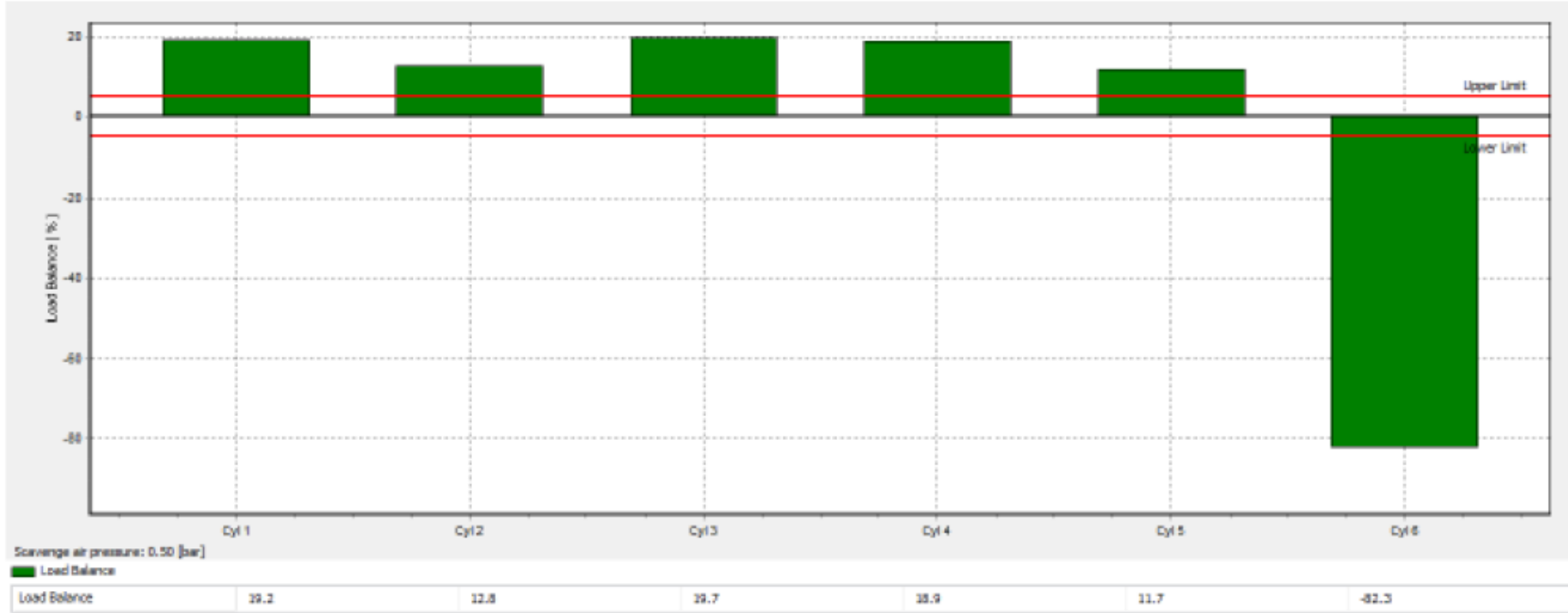


DEUTZ TBD 645 L6

Püskürtme Avansı bozuk silindirde Pcomp ve Pmax değişimi

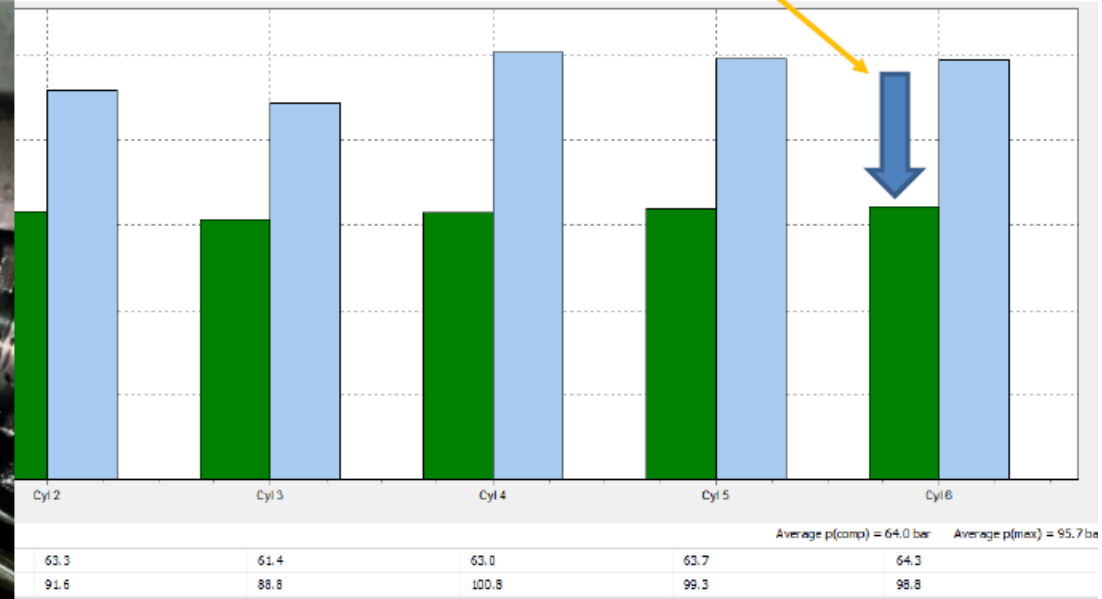
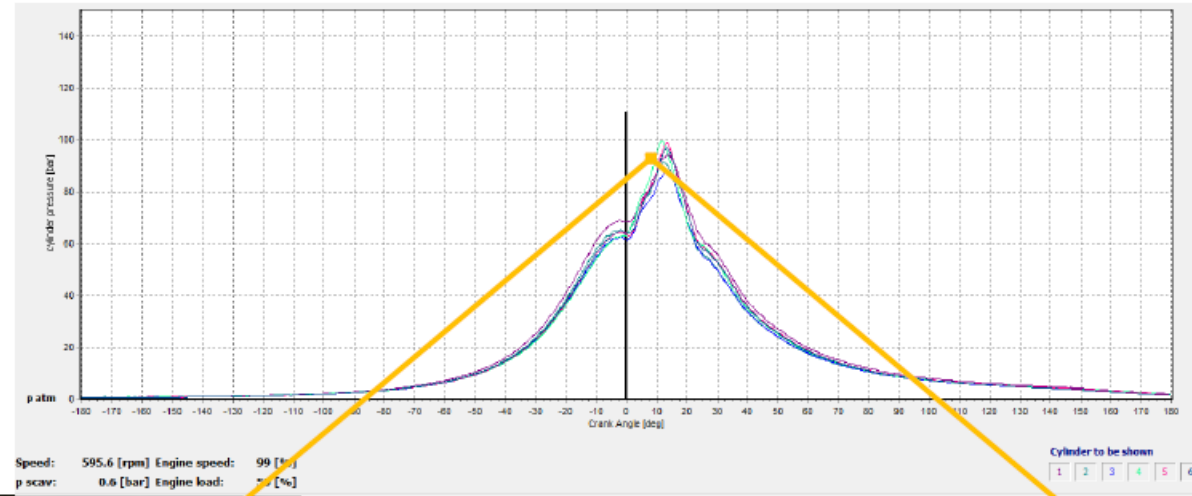


Silindirler arası yük dengesi bozuk ana makine



Silindirlerarası yük dengesi $\pm 5\%$ limitinin çok üzerinde. Krank üzerine binen yük dengesiz.

PÜSKÜRTME AVANS AYARI DÜZELTİLMİŞ ANA MAKİNE



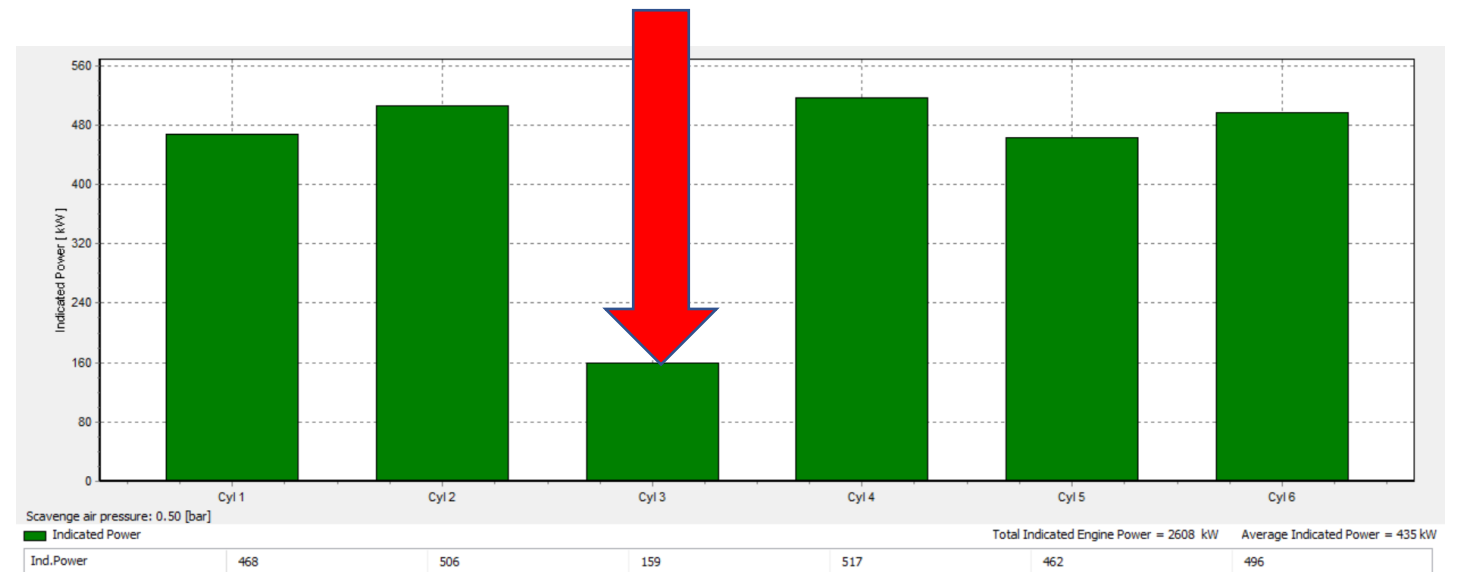
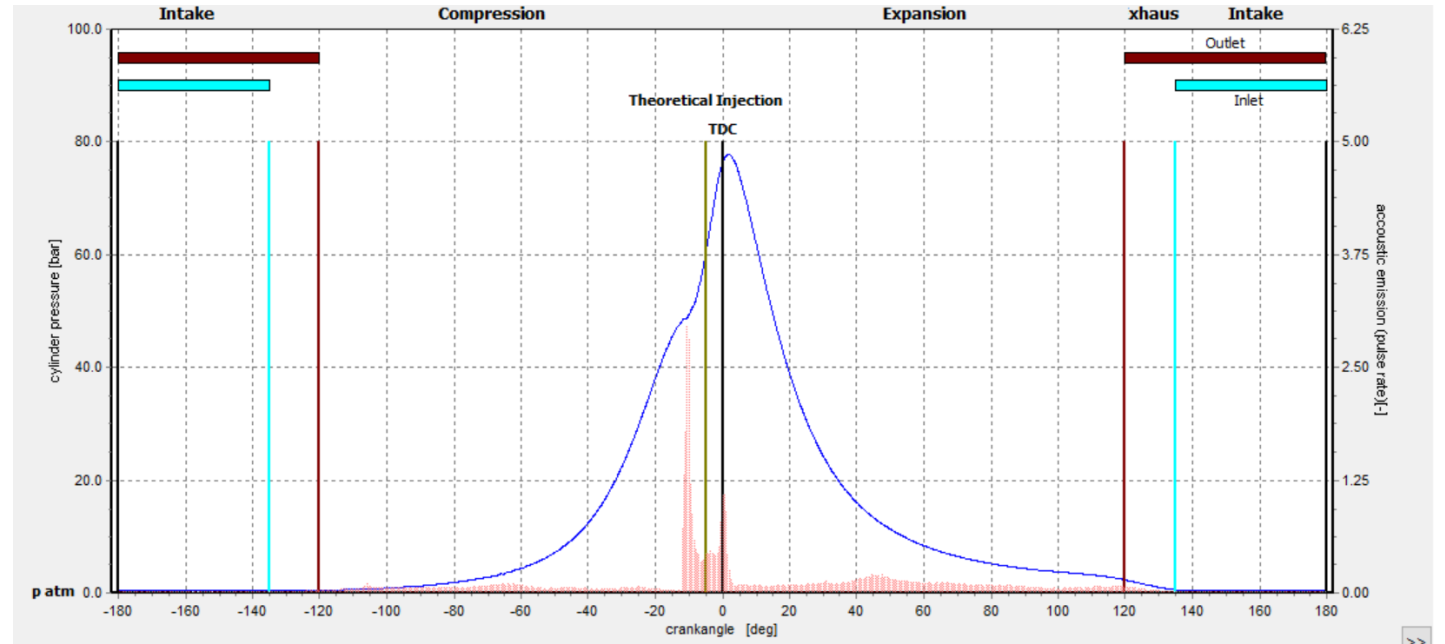
Püskürtme Avansı ve index ayarı yapılmış ana makinenin yük dağılımı

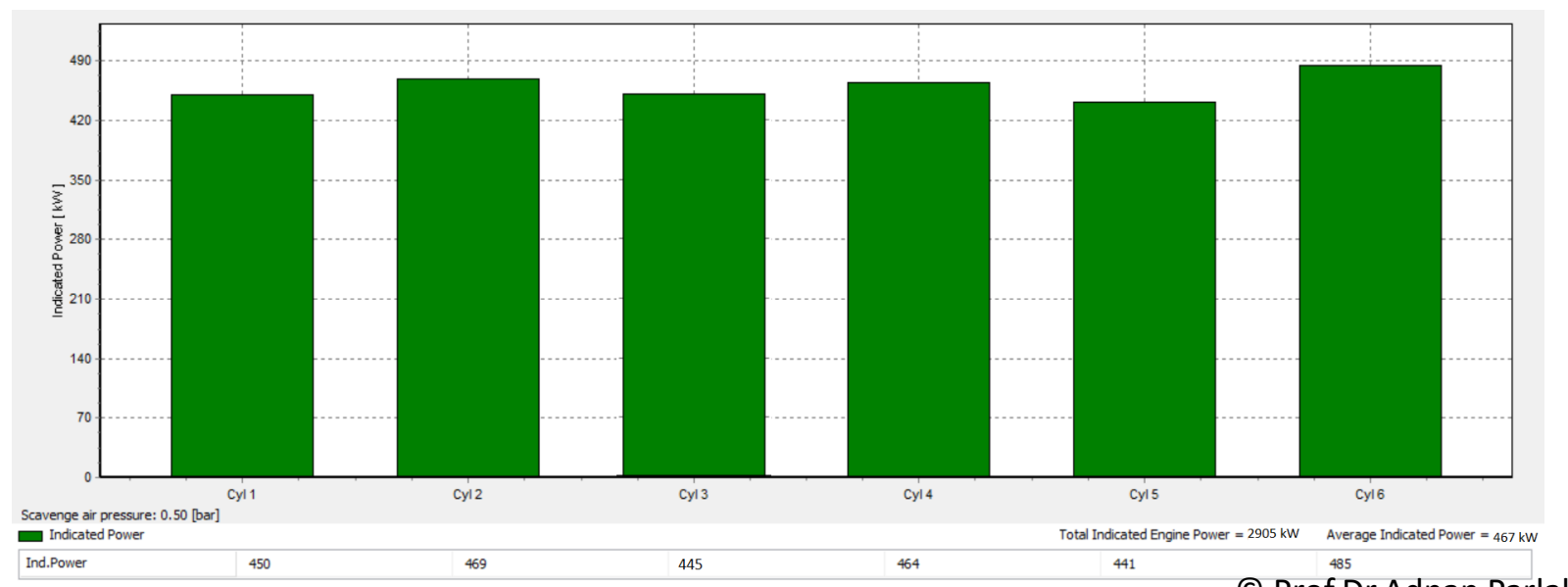
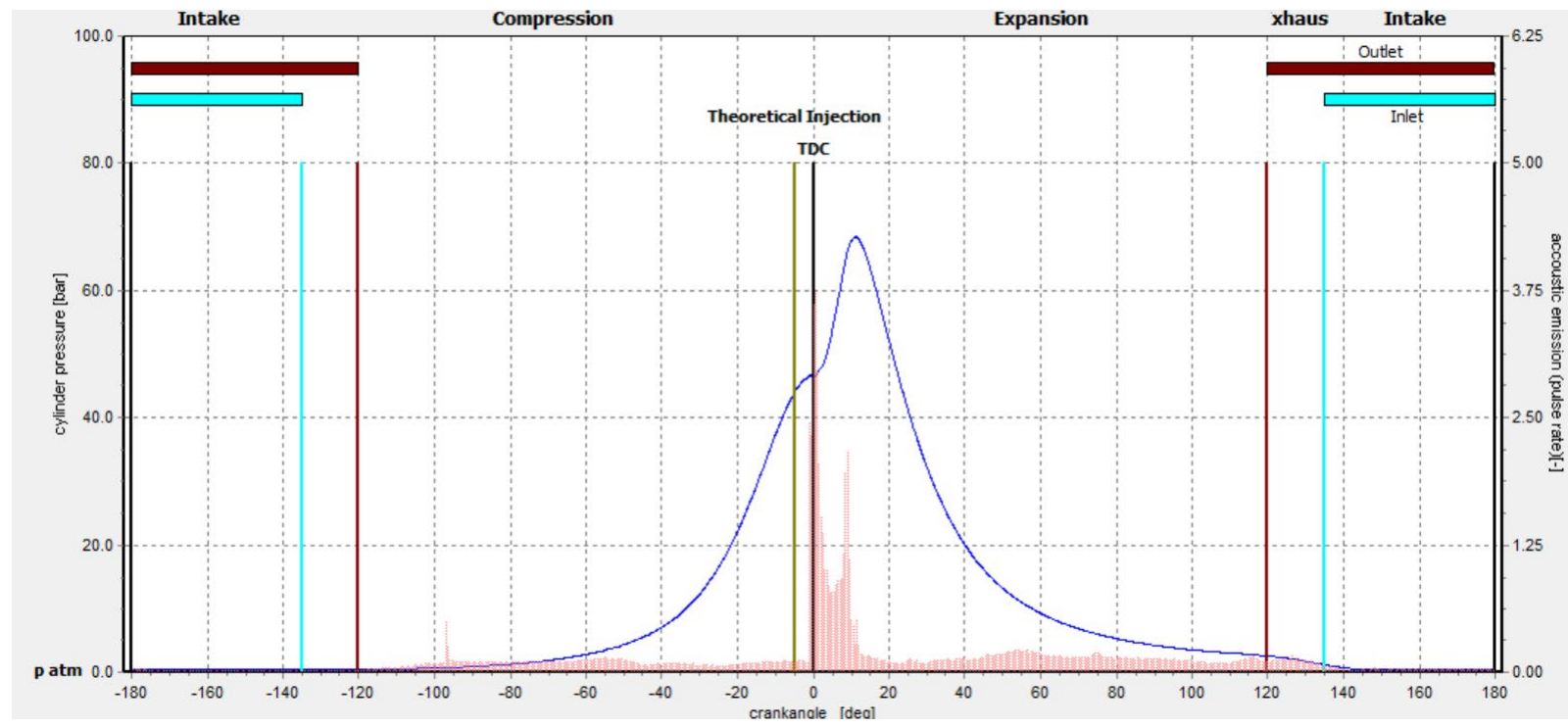
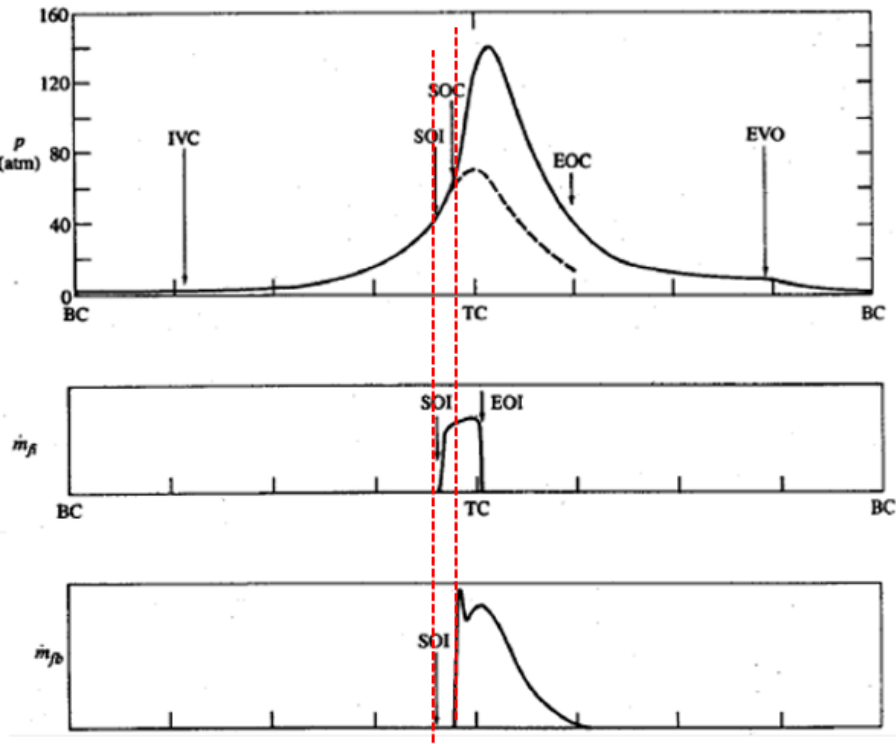


SULZER 6RTA48 ÖLÇÜM DEĞERLERİ



Avans Bozukluğu





TE3231

FOR YOUR REFERENCE

Data Sheet No. 3231110

Summary Data of Shop Trial					
Remarks Temperature of exh. gas at cylinder outlet is measured by ship's thermo-sensor.				Approved	
				Checked	
				Drawn	
Date	May 22, 2000				
Data Sheet No.	3231111	3231112	3231113	3231114	
Load %	50	75	85	100	
Room Temp. °C	18	18	19	19	
Barom. Press. hPa	1014	1014	1014	1014	
Engine Speed rpm	95.1	109.0	113.9	119.9	
Output kW	3637	5379	6074	7170	
Fuel Oil Temp. °C	34	35	35	35	
Specific Fuel Oil Consump. g/kW/Hr	Measured	172.8	170.3	170.4	171.7
	1) LCV correction	171.1	168.7	168.8	170.0
	2) ISO reference	---	---	169.4	---
Pmax. bar	93.2	121.8	132.2	140.2	
Pcomp. bar	71.0	101.5	111.5	127.2	
Pump Mark Index	47.2	57.8	62.2	69.4	
Exh. Gas Temp. Cyl. Outlet °C	310	314	319	339	
Scav. Air	Temp. °C	23	27	29.6	32
	Press. MPa	0.094	0.172	0.20	0.243
Turbo Charger Speed rpm	No. 1	11000	14100	14900	15950
	No. 2	---	---	---	---
	No. 3	---	---	---	---
	No. 4	---	---	---	---
Exh. Gas Temp. T/C Inlet °C	No. 1	330	330	340	370
	No. 2	---	---	---	---
	No. 3	---	---	---	---
	No. 4	---	---	---	---
Exh. Gas Temp. T/C Outlet °C	No. 1	271	238	235	243
	No. 2	---	---	---	---
	No. 3	---	---	---	---
	No. 4	---	---	---	---

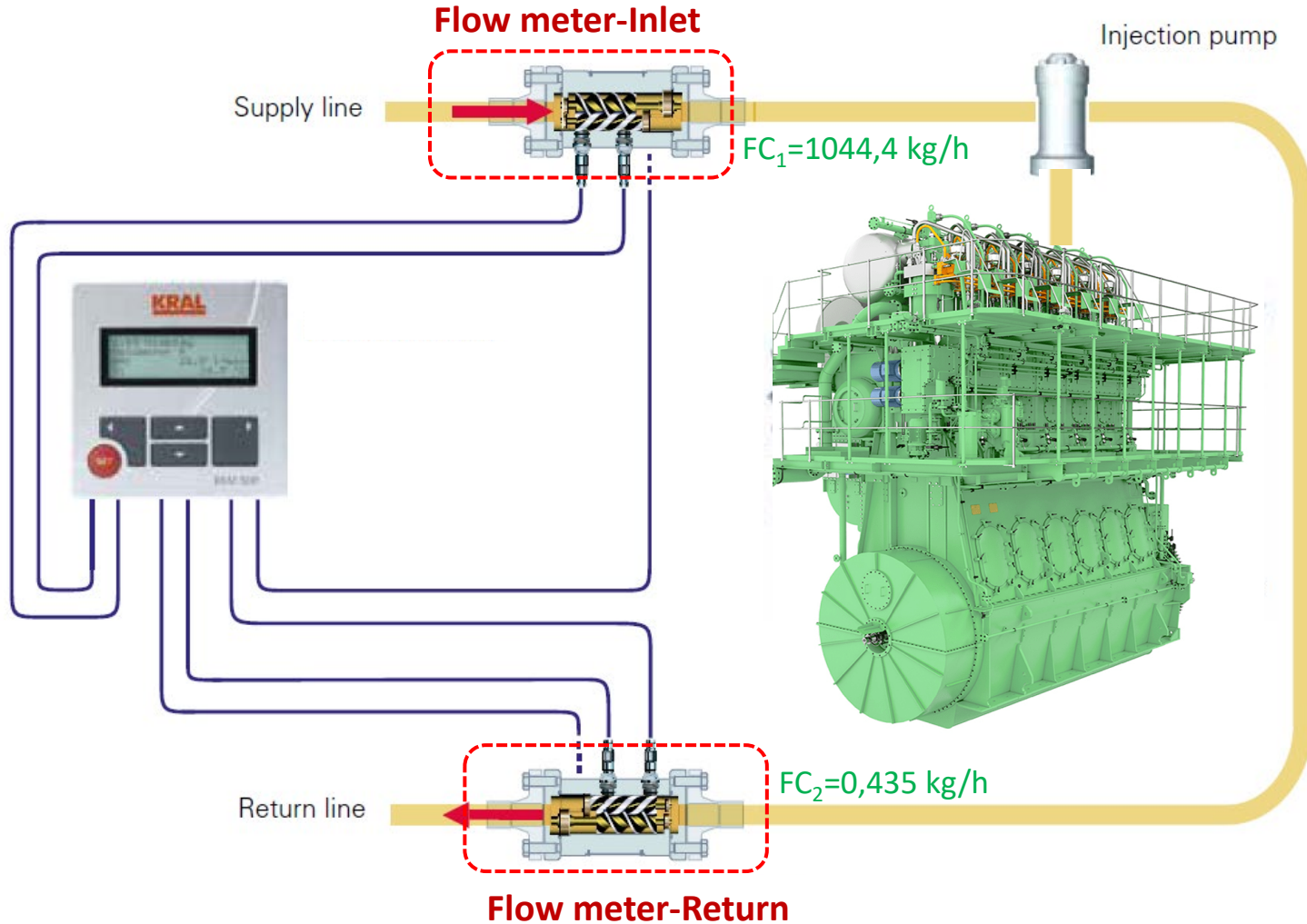
Note : 1) SFOC is corrected to LCV 42700 kJ/kg

2) SFOC is corrected to ISO reference conditions.

1. Geminin 12 knot (Eco Speed) ve 102 rpm'de çalıştığı varsayılmıştır.
2. Alt Isıl Değer (LCV) HSFO için 41.000 kJ/kg ve VLSFO için 42300 kJ/kg olarak alınmıştır.
3. Deniz sıcaklık değerleri www.seatemperature.org adresinden alınmıştır.
4. Hava sıcaklığı değerleri <https://www.meteoblue.com> adresinden alınmıştır.
5. Barometrik basınç 1014 hPa olarak alınmıştır.
6. Shop test değerleri referans alınmıştır. Motordaki bozunum ihmal edilmiştir.



Özgül Yakıt Tüketiminin Hesaplanması



Fuel Consumption :

$$FC = FC_{inlet} - FC_{return}$$

$$FC = 1.044.386 \text{ g / h} - 4350 \text{ g / h} = 1.040.036,4 \text{ g/h}$$

Brake or Effective Power (kW)

$$Power = P_{@85\%} = 6105 \text{ kW}$$

Specific Fuel Consumption (SFC)

$$SFC_{net} = \frac{FC}{P_{@85\%}} = \frac{1.040.036,4 \text{ g/h}}{6105 \text{ kW}} = 170,4 \text{ g/kWh}$$

LCV ye göre SFC Düzeltme

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$$SFC_{LCV} = \frac{SFC_{meas} \times LCV_{test}}{LCV_{std}}$$
$$= \frac{170,4 \text{ g/kWh} \times 42.331 \text{ kJ/kg}}{42.700 \text{ kJ/kg}}$$
$$= 168,4 \text{ g/kWh}$$

Since the Lower Heat Value (LCV) value of the fuel used is lower than the standard value, this means that the main Engine will consume 2 g/kWh more fuel for the same power output at 85% load.

Extra fuel consumption for the same power output of ME:

$$\Delta FC = 2 \text{ g / kWh} * 6105 \text{ kW}$$

$$= +12,2 \text{ kg / h}$$

$$= +293 \text{ kg / day}$$

$$= +8,79 \text{ MT / Month}$$

6000h working condition

$$= 12,2 (\text{kg / h}) * 6000 (\text{h / Year}) / 1000$$

$$= +73,2 \text{ MT/Year}$$

Extra cost

$$= 73,2 \text{ MT / Year} \times 500 \$ / \text{MT} = 36,600 \$$$

If the LCV value is higher than the standard value, then SFC:

$$= \frac{168,4 \text{ g/kWh} \times 42.700 \text{ kJ/kg}}{43.500 \text{ kJ/kg}}$$
$$= 165,3 \text{ g/kWh}$$

$$\Delta FC = 5,1 \text{ g / kWh} * 6105 \text{ kW}$$

$$= -31,1 \text{ kg / h}$$

$$= -506 \text{ kg / day}$$

$$= -15,2 \text{ MT / Month}$$

6000h working condition

$$= -31,1 (\text{kg / h}) * 6000 (\text{h / Year}) / 1000$$

$$= -186,6 \text{ MT/Year}$$

Saving

$$= -186,6 \text{ MT / Year} \times 500 \$ / \text{MT} = 93,300 \$$$

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ISO şartlarına göre SFC düzeltme (ISO SFC)



Parameters	Reference Condition	Conversion Factors
Room Temp.	25 °C	+0,2% /10 °C Rise
Barometric Pres.	1000 hbar	-0,02% /10 hPa Rise
Air Cool.Water Temp.	25 °C	+0,6% /10 °C Rise
Fuel Oil Lower Cal.Value (LCV)	42.700 kJ/kg	-1,0% /1% Rise

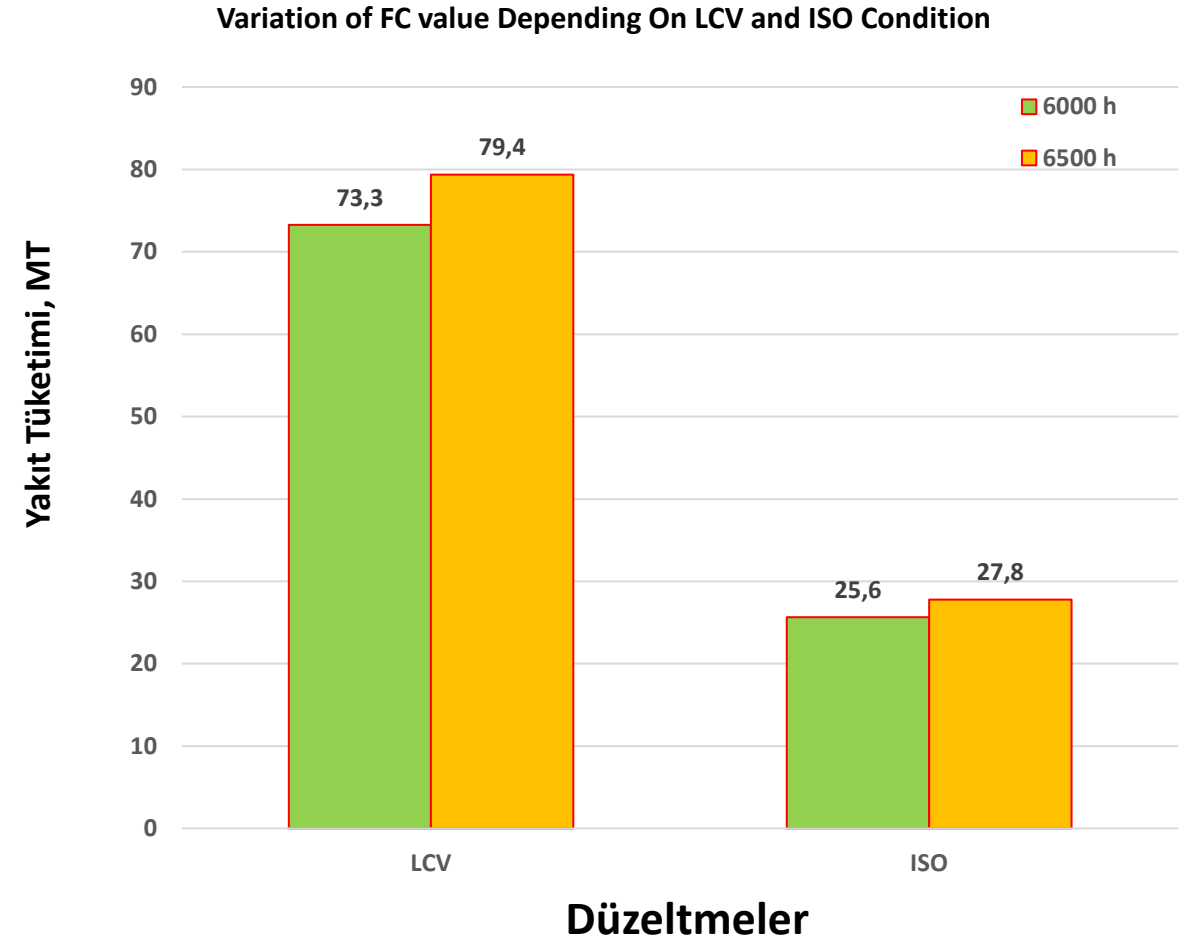
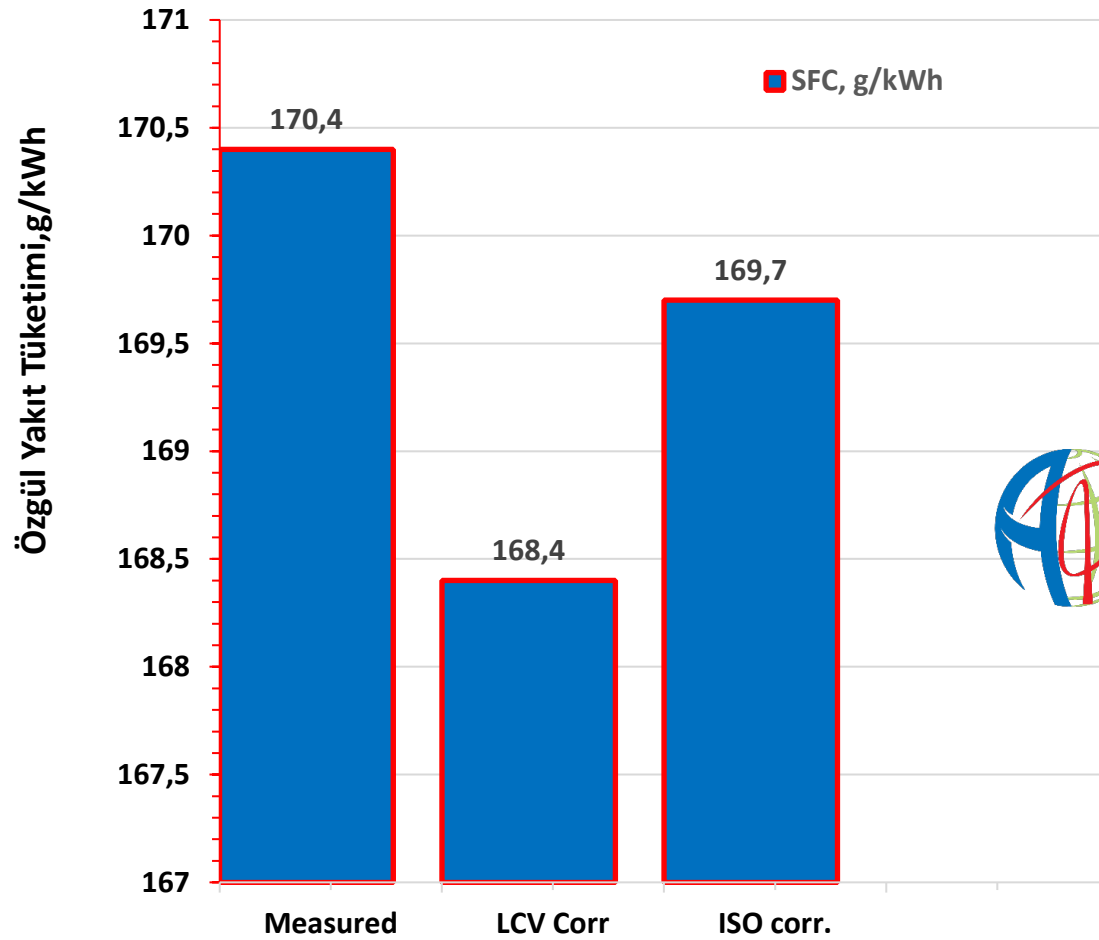
$$SFC_{ISO} = SFC_{meas} \times \left(\frac{(25 - T_o) \times 0,2 + (1000 - P_o) \times (-0,02) + (25 - T_c) \times 0,6}{1000} + 1 \right) \times LCV_{meas} / 42.700$$

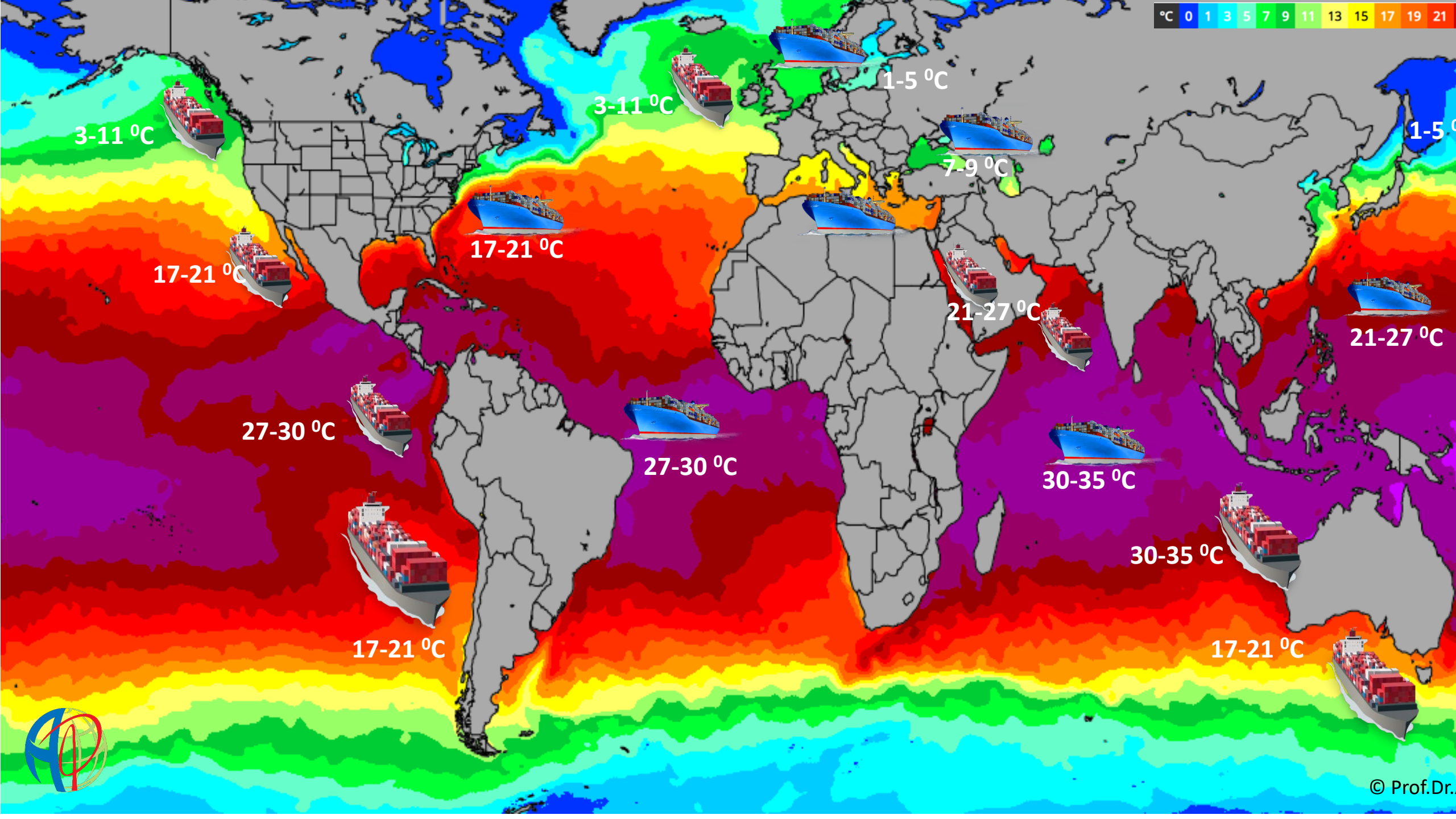
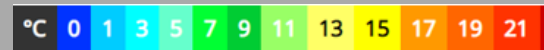
Servis Condition	ISO Condition
$T_o = 10,3 \text{ °C}$	$T_o = 25 \text{ °C}$
$P_o = 1024 \text{ hPa}$	$P_o = 1000 \text{ hPa}$
$T_c = 22,7 \text{ °C}$	$T_c = 25 \text{ °C}$

$$SFC_{ISO} = 170,4 \text{ g/kWh} \times \left(\frac{(25 - 10,3) \times 0,2 + (1000 - 1024) \times (-0,02) + (25 - 22,7) \times 0,6}{1000} + 1 \right) \times 42.331 \text{ kJ/kg} / 42.700 \text{ kJ/kg}$$

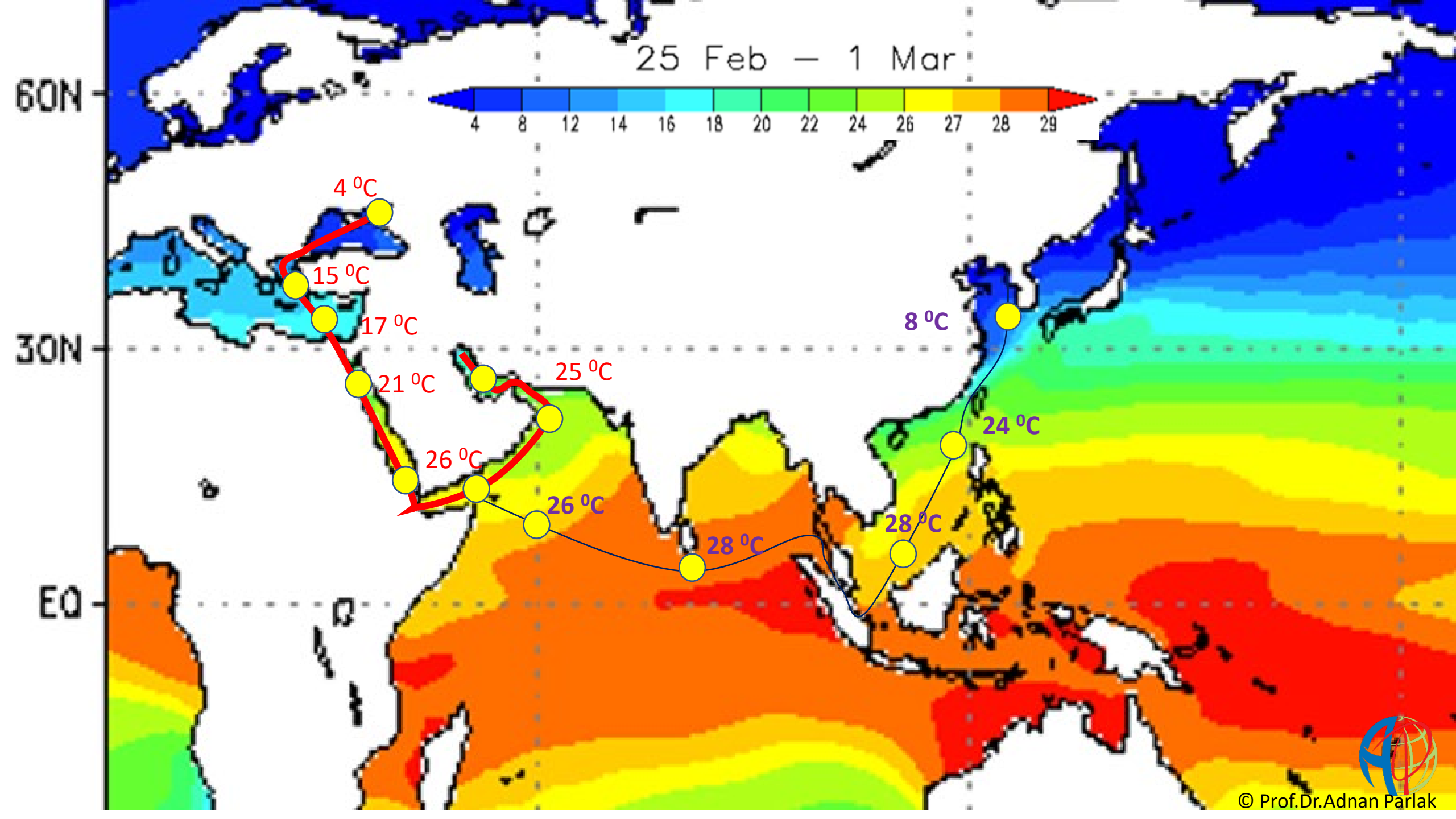
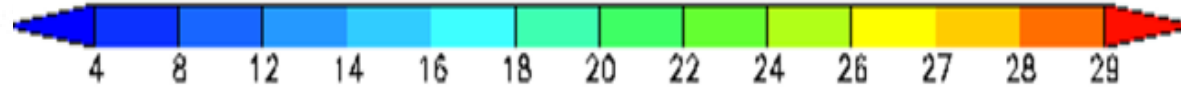
$$= 169,7 \text{ g/kWh}$$

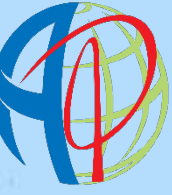
Çevresel şartlara ve LCV nin SFC ve FC üzerine etkileri





25 Feb - 1 Mar



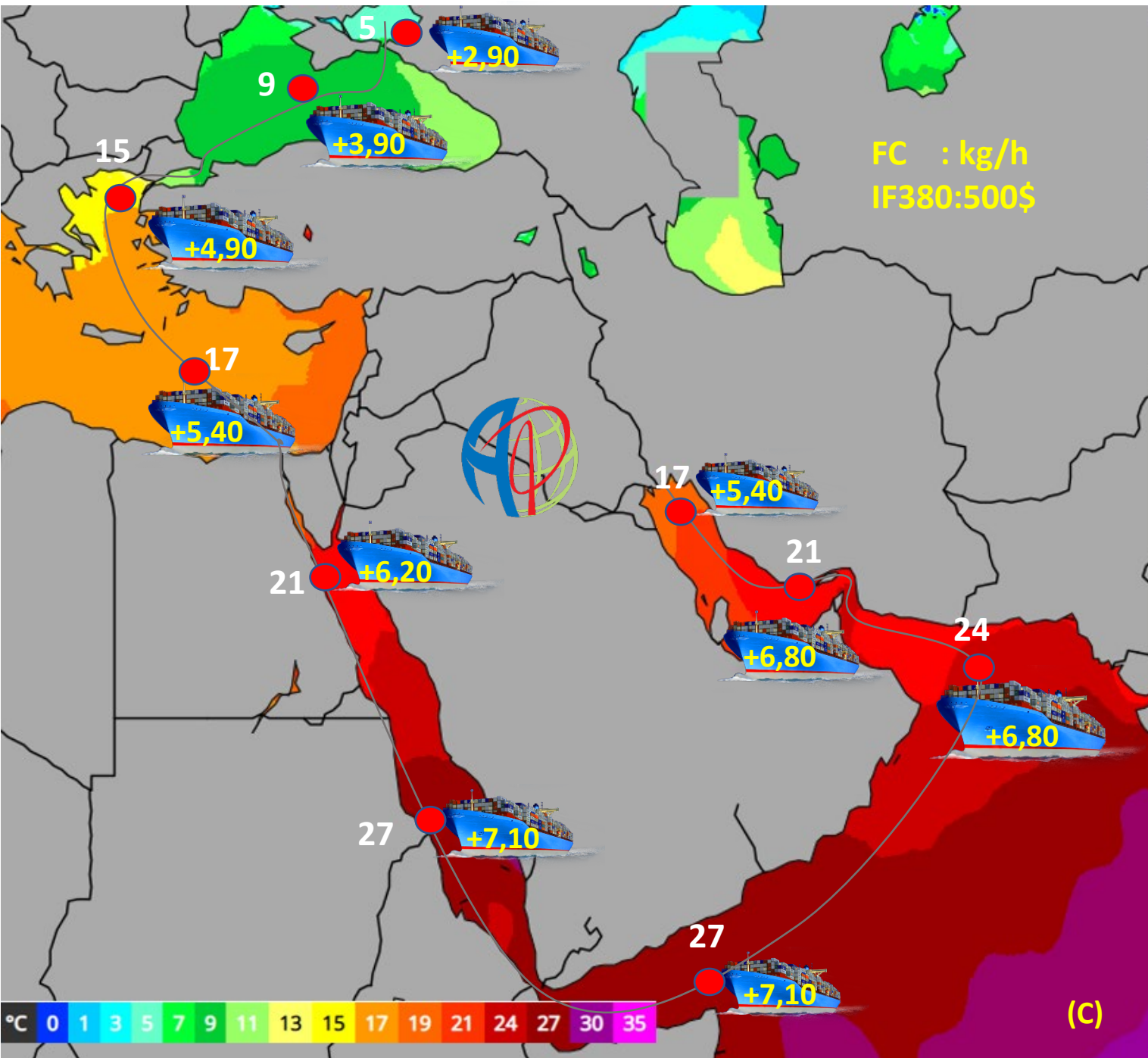


▪ **Çevre Şartları ve Yakıt Isıl Değerinin Yakıt Tüketimi Üzerine Etkileri**

Çevre şartlarına göre Özgül yakıt tüketimindeki değişim

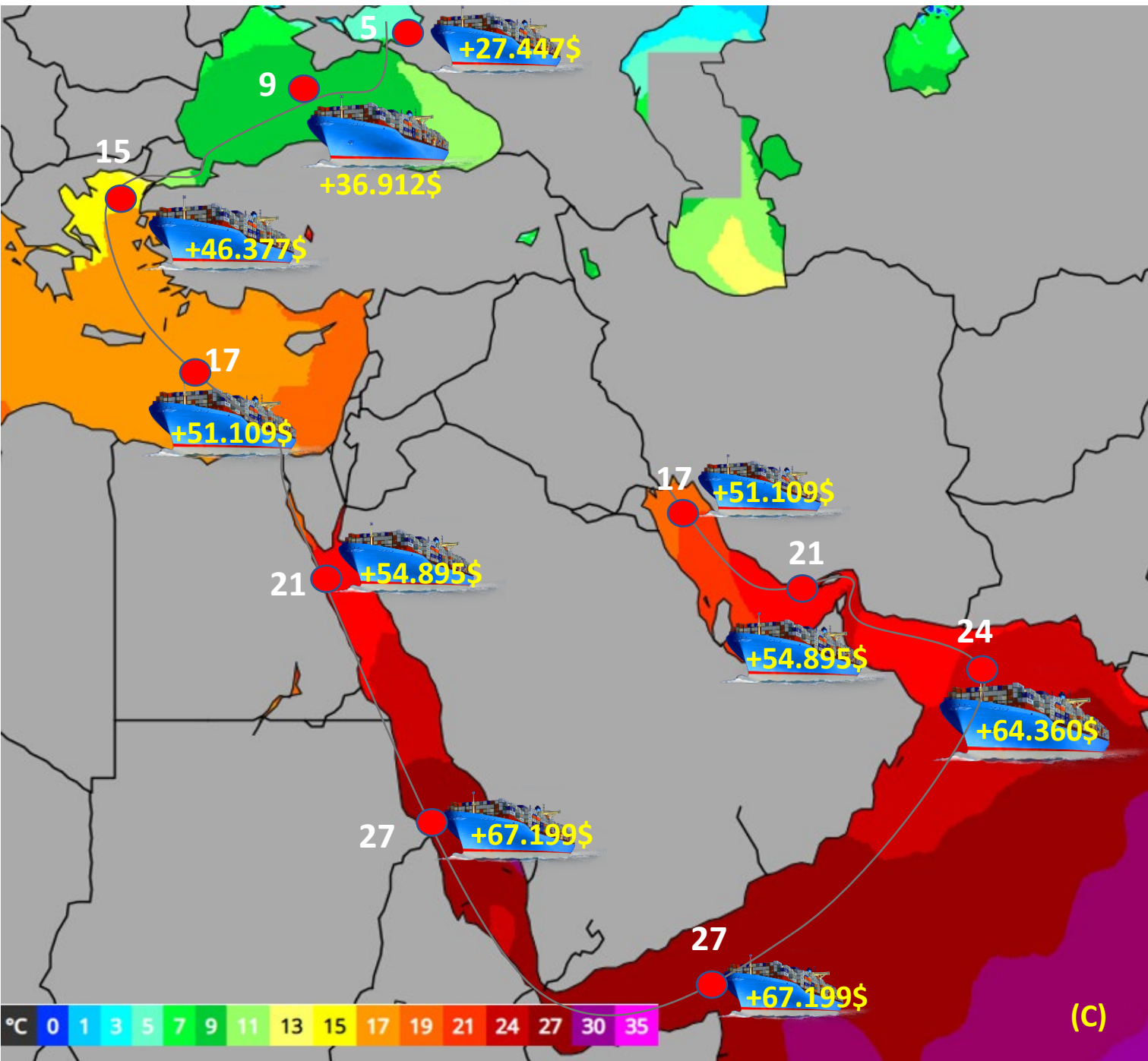


LCV :41.000 kJ/kg.



Çevre şartlarına göre yakıt tüketim maliyeti

LCV : 41.000 kJ/kg.



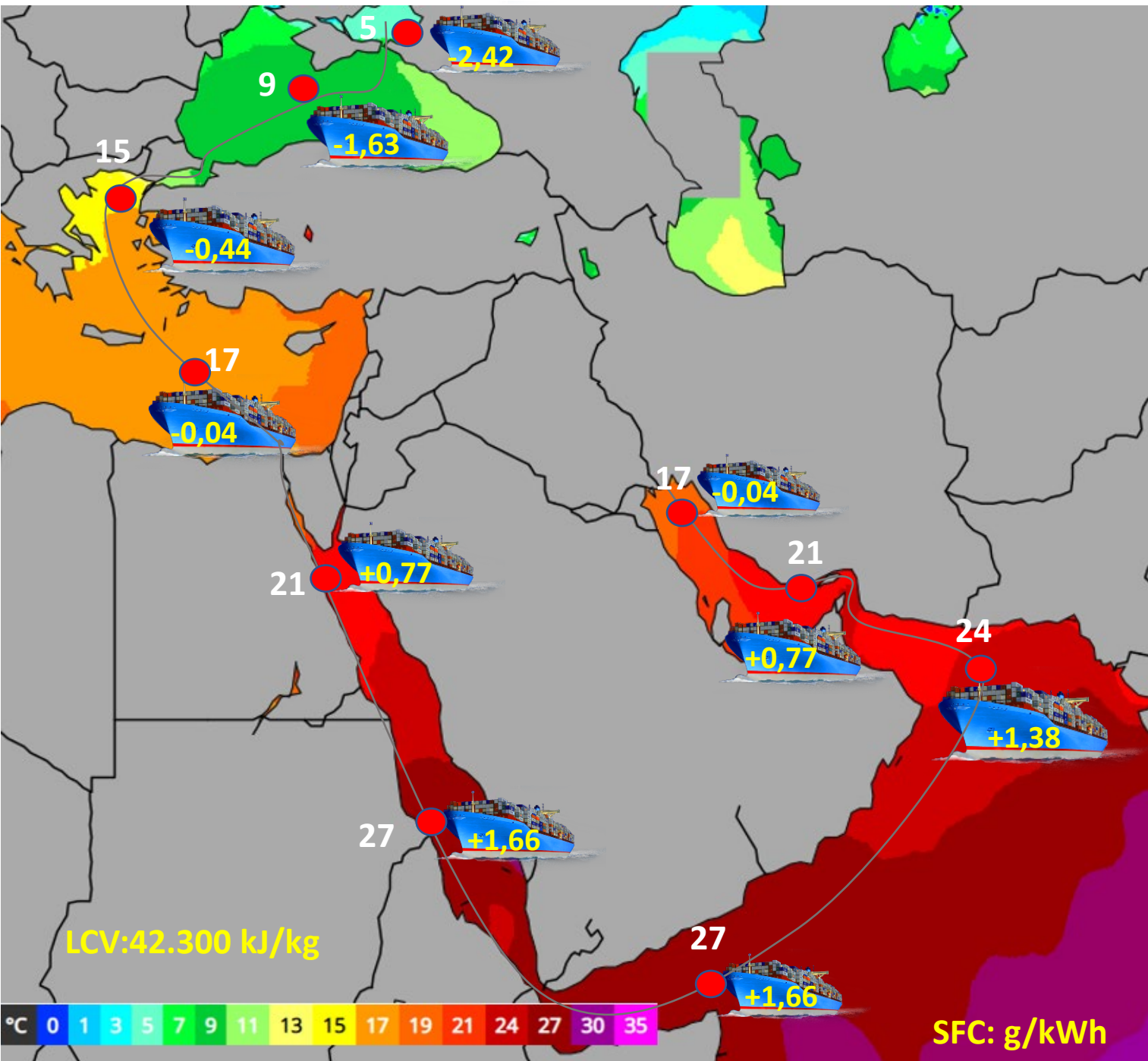
Deniz suyu	Hava Sıcaklığı	LCV:41.000 kJ/kg				AFC.kg/h		
		SFC_LCV	FC_ISO	ΔSFC	ΔAFC.kg/h	6000h	6500h	7000h
5	5	172,90	18,70	2,90	13,07	27.447,6	29.734,9	32.022,2
9	9	173,90	18,80	3,90	17,58	36.912,3	39.988,4	43.064,4
15	15	174,90	18,90	4,90	22,08	46.377,0	50.241,8	54.106,5
17	17	175,40	18,97	5,40	24,34	51.109,4	55.368,5	59.627,6
19	19	175,80	19,01	5,80	26,14	54.895,3	59.469,9	64.044,5
21	21	176,20	19,06	6,20	27,94	58.681,1	63.571,2	68.461,3
24	24	176,80	19,13	6,80	30,65	64.360,0	69.723,3	75.086,6
27	25	177,10	19,16	7,10	32,00	67.199,4	72.799,3	78.399,3
30	26	177,60	19,21	7,60	34,25	71.931,7	77.926,0	83.920,3
35	28	177,60	19,25	7,60	34,25	71.931,7	77.926,0	83.920,3

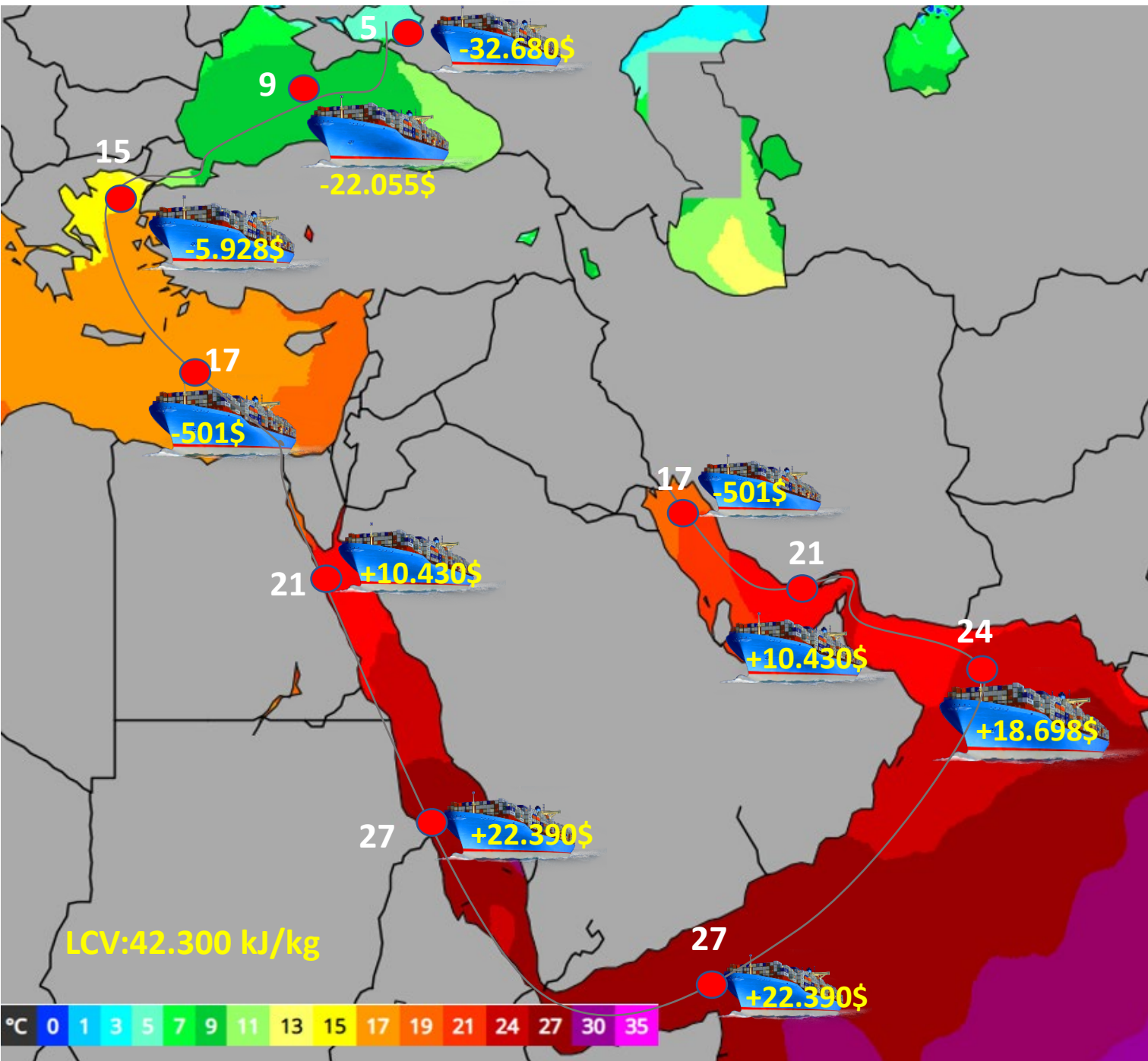


Çevre Şartları ve LCV nin Yakıt tüketimi üzerine etkileri

Çevresel Şartlarının SFC üzerine etkileri

LCV: 42.300 kJ/kg.





Variation of FC According to Environment Conditions in case of VLSFO Usage.



LCV of HSFO is taken as 42.300 kJ/kg. The figures can show some changes according to LCV



Deniz suyu	Hava Sıcaklığı	LCV:42.300 kJ/kg				ΔFC,kg/h		
		SFC_LCV	FC_ISO	ΔSFC	ΔFC,kg/h	6000h	6500h	7000h
5	5	167,58	18,13	-2,42	-10,89	-32.680,6	-35.404,0	-38.127,4
9	9	168,37	18,21	-1,63	-7,35	-22.055,0	-23.892,9	-25.730,8
15	15	169,56	18,34	-0,44	-1,98	-5.927,7	-6.421,6	-6.915,6
17	17	169,96	18,38	-0,04	-0,17	-500,8	-542,5	-584,2
19	19	170,37	18,43	0,37	1,65	4.952,0	5.364,6	5.777,3
21	21	170,77	18,47	0,77	3,48	10.430,7	11.300,0	12.169,2
24	24	171,38	18,54	1,38	6,23	18.698,2	20.256,3	21.814,5
27	25	171,66	18,57	1,66	7,46	22.389,8	24.255,7	26.121,5
30	26	171,93	18,60	1,93	8,70	26.093,4	28.267,8	30.442,3
35	28	172,45	18,65	2,45	11,02	33.070,3	35.826,1	38.582,0

Date	:	2008-08-12
Ship owner	:	COSCO GROUP
Ship yard	:	COSCO ZHOUSHAN SHIPYARD
Ship No.	:	ZS07035
Engine Type	:	6S50MC-C7
Engine No.	:	SB6S50-7161
Engine output	:	9,480 kW
Engine speed	:	127 rpm
Fuel oil specific gravity at 15 °C	:	0.8660
Fuel oil lower calorific value	:	10,152 kcal/kg

Engine load (%)	25%	50%	75%	90%	100%	100%	110%
Engine output (kW)	2,370 kW	4,740 kW	7,110 kW	8,532 kW	9,480 kW	9,480 kW	10,428 kW
Measuring quantity (kg), AL	74.1 kg	140.5 kg	206.4 kg	248.5 kg	278.4 kg	277.1 kg	310.5 kg
Measuring time (s), B	600 s	600 s	600 s	600 s	600 s	600 s	600 s
Specific fuel oil consumption (Measured, g/kW-h), B3=(ALX3600X1000)/(B3XkW)	187.59	177.85	174.18	174.75	176.20	175.38	178.65
Ambient temperature (°C), T1	29.0 °C	29.0 °C	28.0 °C	30.0 °C	31.0 °C	31.0 °C	31.0 °C
Charge air coolant temperature (°C), T2	29.0 °C	30.0 °C	30.0 °C	31.0 °C	30.0 °C	30.0 °C	30.0 °C
Ambient pressure (mbar), P	1013 mbar	1013 mbar	1013 mbar	1013 mbar	1011 mbar	1013 mbar	1010 mbar
Conversion factor by ambient temperature, T1F=0.002X(25-T1)/10	-0.00080	-0.00080	-0.00060	-0.00100	-0.00120	-0.00120	-0.00120
Conversion factor by boost Air coolant temperature, T2F=0.006X(25-T2)/10	-0.00240	-0.00300	-0.00300	-0.00360	-0.00300	-0.00300	-0.00300
Conversion factor by ambient pressure, PF=0.0002X(P-1000)/10	0.00026	0.00026	0.00026	0.00026	0.00022	0.00026	0.00020
Conversion factor by lower calorific value, HF=(H-10200)/10200	-0.00470588	-0.00470588	-0.00470588	-0.00470588	-0.00470588	-0.00470588	-0.00470588
Total conversion factor, TCF=T1F+T2F+PF+HF	-0.00765	-0.00825	-0.00805	-0.00905	-0.00869	-0.00865	-0.00871
Converted value of F.O consumption based on ISO (g/kW-h), B1=B3+(B3*TCF)	186.16	176.38	172.78	173.17	174.67	173.86	177.10
Guarantee of fuel oil consumption at CSR (Load : 90%)	168.9 g/kW-h + 5.0% = (177.3g)						

CALCULATION for SPECIFIC FUEL OIL CONSUMPTION (S.F.O.C.)

At 85% Load

CALCULATION FORMULA for MEASURED S.F.O.C.

$$\begin{aligned} \text{S.F.O.C. (MEAS.)} &= \text{F.O.WEIGHT(kg)} \times 60 \times 60 \times 1000 / \text{TIME(sec.)} / \text{OUTPUT} \\ \text{F.O. DRAIN} &= \text{F.O.DRAIN AMOUNT(g)} \times 60 \times 60 / \text{TIME(sec.)} / \text{OUTPUT} \\ \text{S.F.O.C. (NET)} &= \text{S.F.O.C. (MEAS.)} - \text{F.O. DRAIN} \end{aligned}$$

Parameter	Measured weight	Measured time
S.F.O.C.(MEAS.)	120 (kg)	408.14 (sec)
F.O. drain	1260 (g)	360 (sec)

$$\begin{aligned} \text{S.F.O.C. (MEAS.)} &= 120 \times 60 \times 60 \times 1000 / 408.14 / 6467 = 163.7 \text{ (g/kW-h)} \\ \text{F.O. DRAIN} &= 1260 \times 60 \times 60 / 360 / 6467 = 1.9 \text{ (g/kW-h)} \\ \text{S.F.O.C. (NET)} &= 163.7 - 1.9 = 161.8 \text{ (g/kW-h)} \end{aligned}$$

CALCULATION FORMULA for S.F.O.C.CORRECTED to ISO CONDITION

$$\text{S.F.O.C. (ISO)} = \text{S.F.O.C. (NET)} \times \left[\frac{(25 - T_o) \times 0.2 + (1000 - P_o) \times (-0.02) + (25 - T_c) \times 0.6}{1000} + 1 \right] \times \text{LCV} / 42700$$

Parameter	ISO condition	Conversion factors	Measured values
To : T/C intake air temp.	25 (°C)	+0.2% / 10°C Rise	17.8 (°C)
Po : Barometric press.	1000 (hPa)	-0.02% / 10hPa Rise	1017.0 (hPa)
Tc : A/C water inlet temp.	25 (°C)	+0.6% / 10°C Rise	22.2 (°C)
LCV : F.O. Lower Calorific Value	42700 (kJ/kg)	-1% / 1% Rise	42070 (kJ/kg)

$$\begin{aligned} \text{S.F.O.C. (ISO)} &= 161.8 \times \left[\frac{(25 - 17.8) \times 0.2 + (1000 - 1017) \times (-0.02) + (25 - 22.2) \times 0.6}{1000} + 1 \right] \times 42070 / 42700 \\ &= 160.0 \text{ (g/kW-h)} \end{aligned}$$

* Guaranteed value at ISO condition is 160.5 (g/kW-h) + 5% (= 168.52 g/kW-h)

Egzoz Sıcaklığının Çevre Şartlarına göre düzeltilmesi

CALCULATION FORMULA for TURBO CHARGER Exh. GAS OUTLET TEMP. CORRECTED to DESIGN CONDITION

At C.S.O. Load

$$\text{T/C Exh. GAS OUT TEMP. (CORRECTED)} = (273 + T_{\text{exh.}}) \times \left(\frac{(25 - T_o) \times 3.16 + (25 - T_c) \times 0.18}{1000} + 1 \right) - 273$$

Parameter	Measured values	Design values	Corrected value
T _{exh.} : T/C Exh. gas outlet temp.	205 (°C)	218 (°C) ± 15°C	216 (°C)
T _o : T/C intake air temp.	17.8 (°C)	25 (°C)	
T _c : A/C water inlet temp.	22.2 (°C)	25 (°C)	

$$\begin{aligned} \text{T/C Exh. GAS OUT TEMP. (CORRECTED)} &= (273 + 205) \times \left(\frac{(25 - 17.8) \times 3.16 + (25 - 22.2) \times 0.18}{1000} + 1 \right) - 273 \\ &= 216 \text{ } ^\circ\text{C} \end{aligned}$$